



(10) **Patent No.:** US 6,848,335 B1
(45) **Date of Patent:** *Feb. 1, 2005

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This patent is subject to a terminal disclaimer.

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Primary Examiner—Vinh T. Luong
(74) Attorney, Agent, or Firm—Shinju Global IP
Counselors, LLP

- (57) **ABSTRACT**

A bicycle shift operating device basically includes first operating member operatively coupled to a winding mechanism by a follower link and an operating link. A second operating member is also preferably operatively coupled to the winding mechanism. The cable winding mechanism is mounted on a first pivot axle and is configured to be selectively retained in a plurality of shift positions. In one shift operating device, the operating link is pivotally mounted on the first pivot axle. In an alternate shift operating device, the operating link is pivotally mounted on a second pivot axle that is spaced from the first pivot axle and parallel to the first pivot axle. The first operating member is preferably arranged to reciprocate in a first direction. The second operating member is preferably arranged to move in a second direction that is substantially parallel to the first direction.

10 Claims, 28 Drawing Sheets

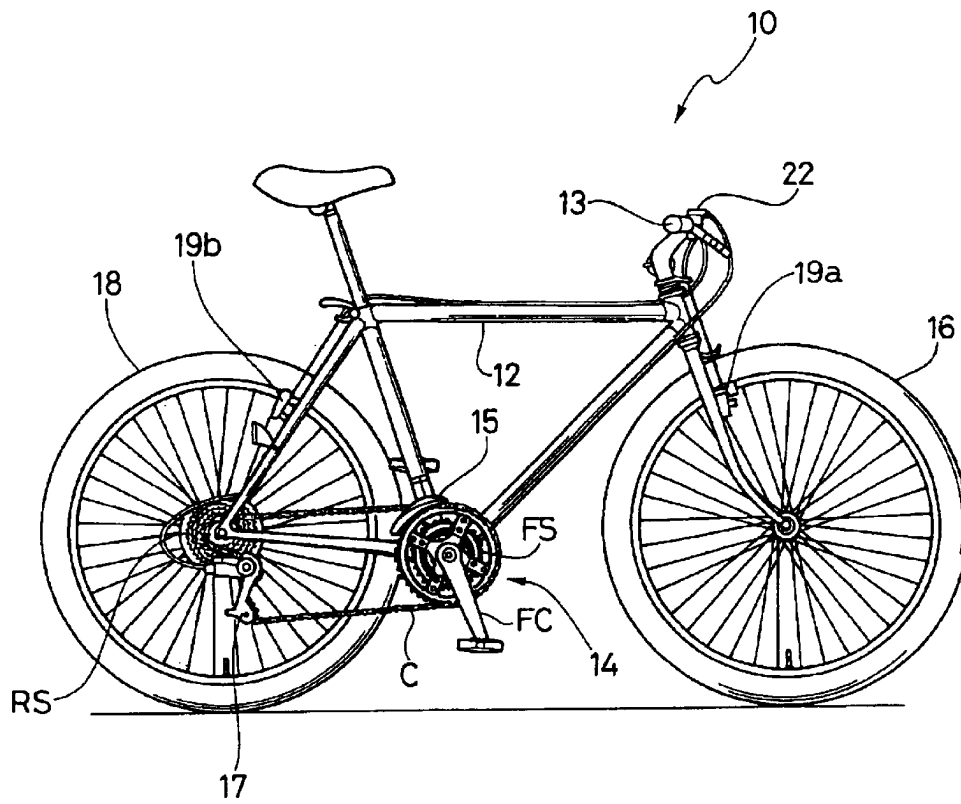
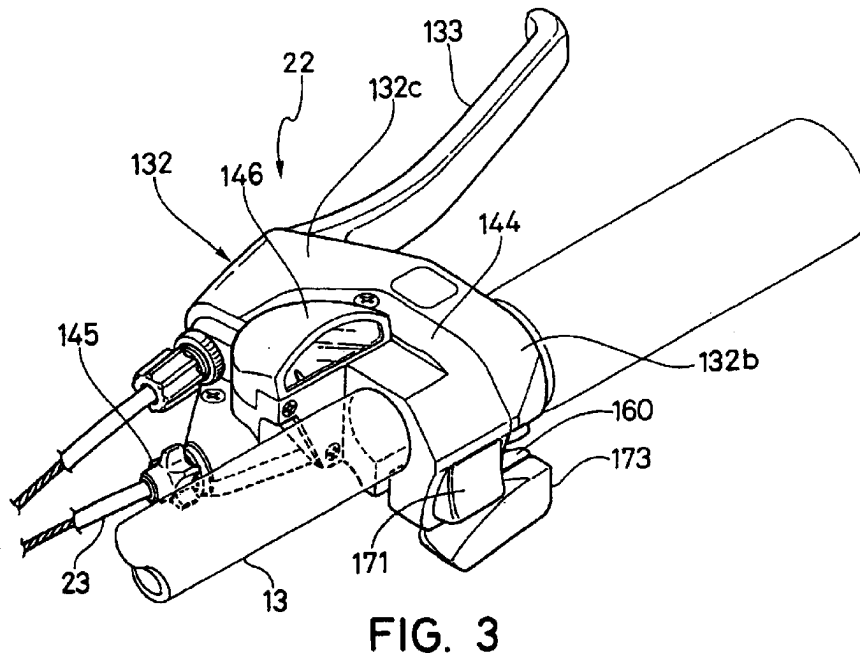
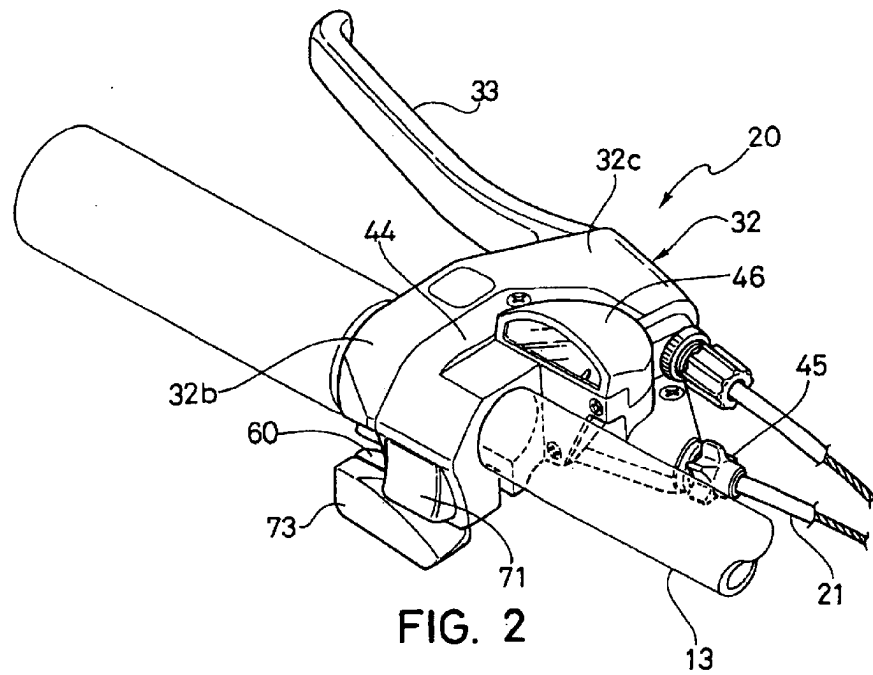


FIG. 1



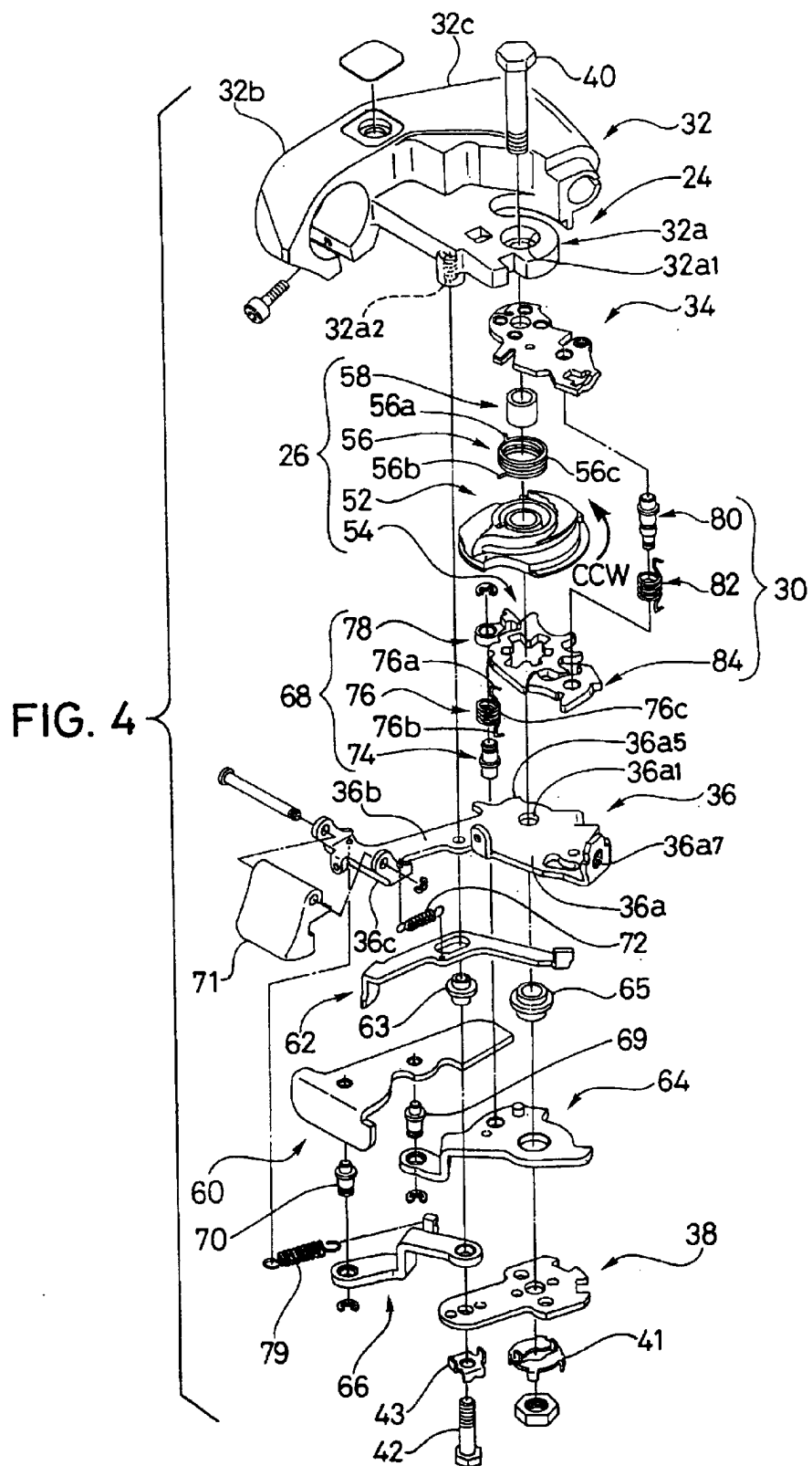
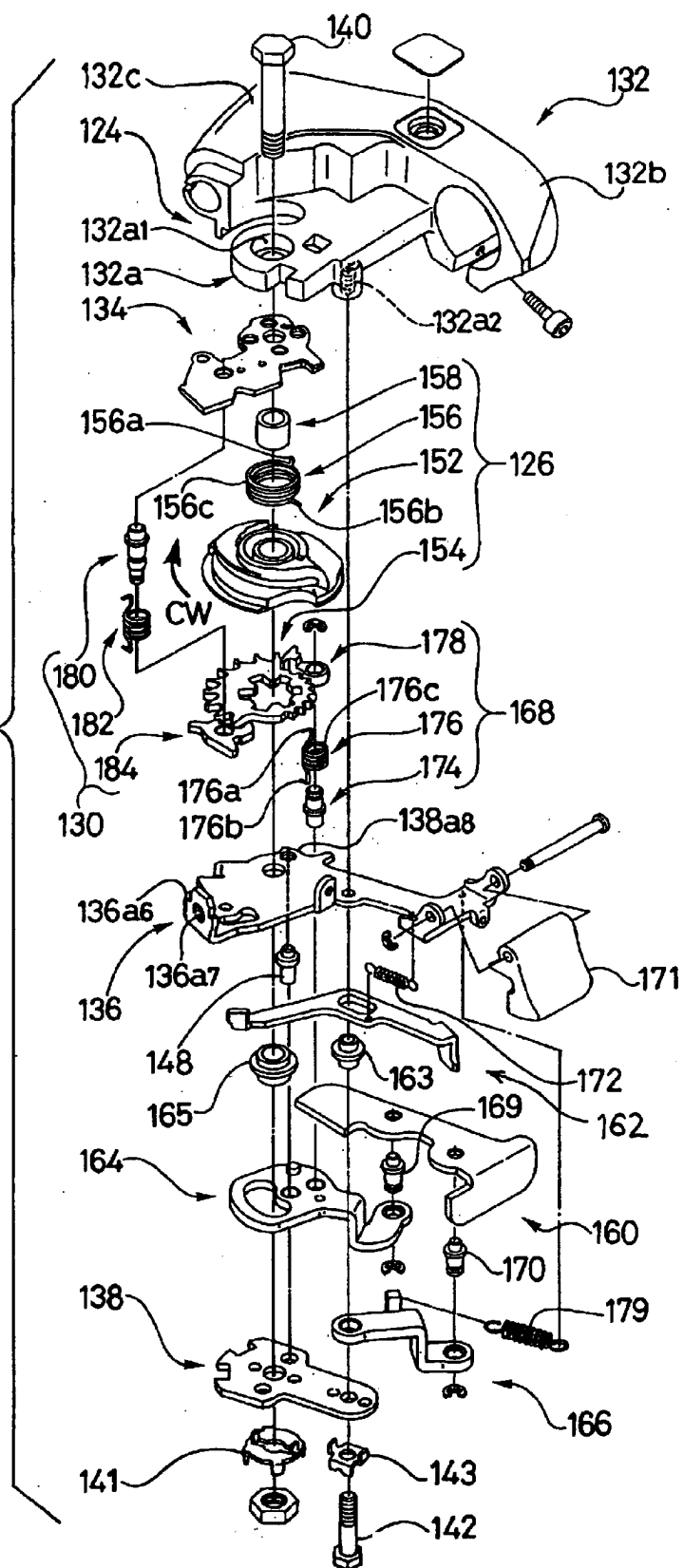


FIG. 5



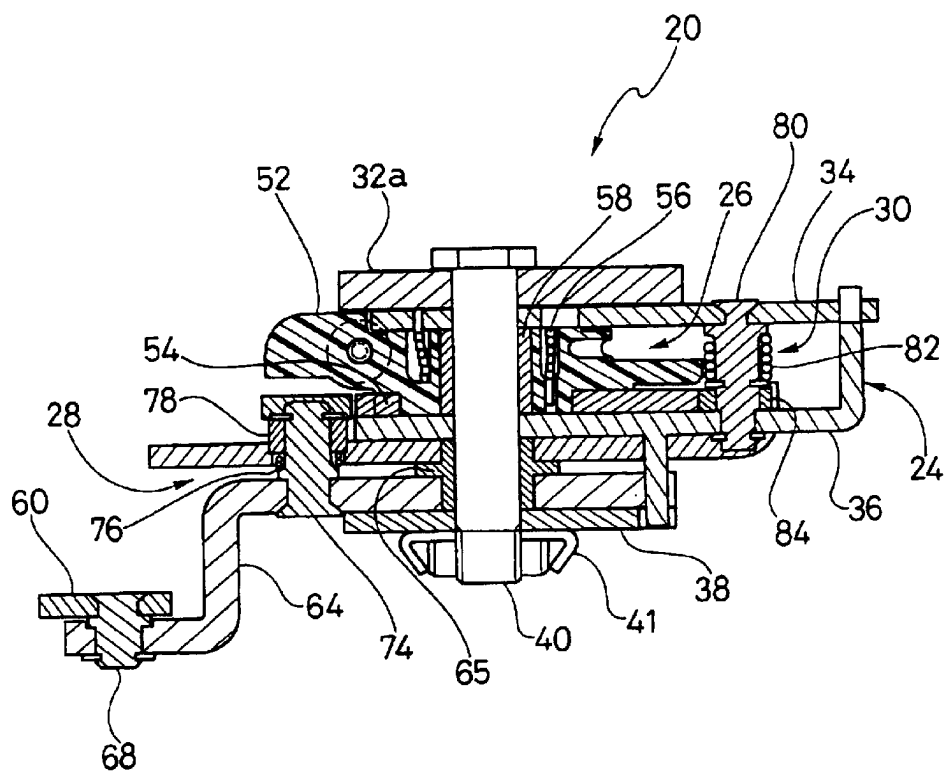
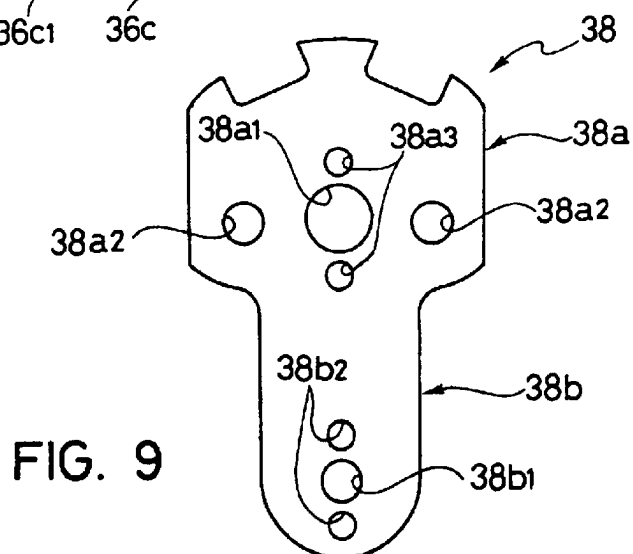
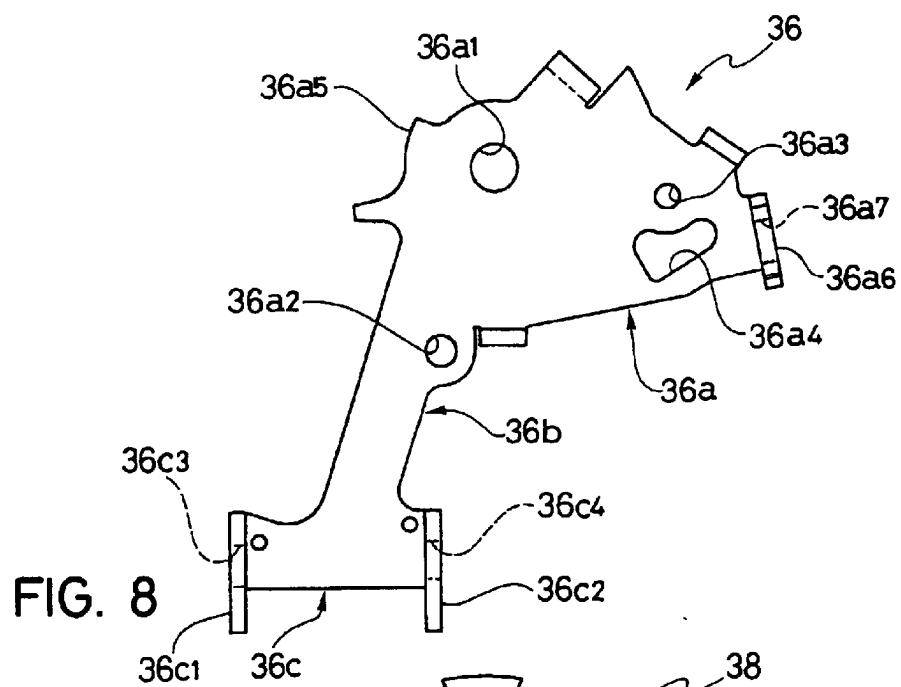
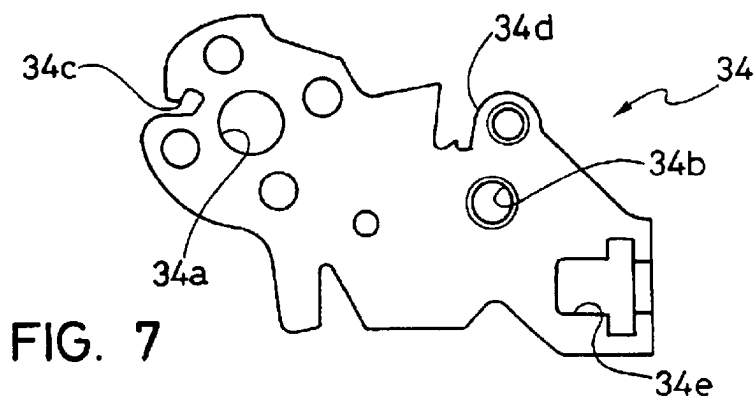


FIG. 6



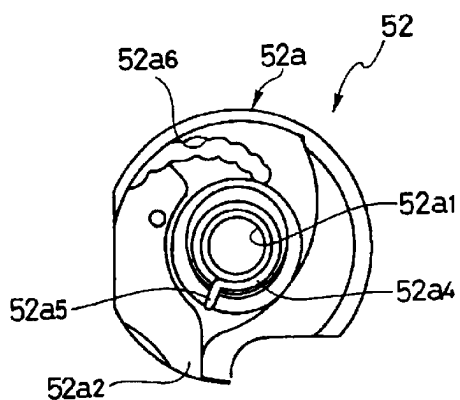


FIG. 10

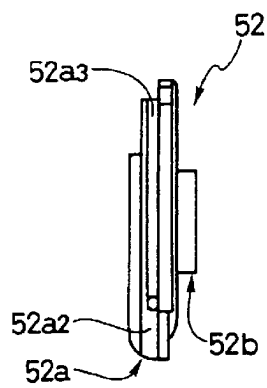


FIG. 11

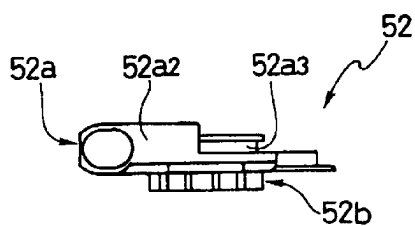


FIG. 12

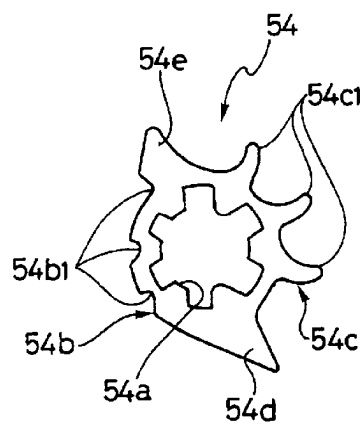


FIG. 14

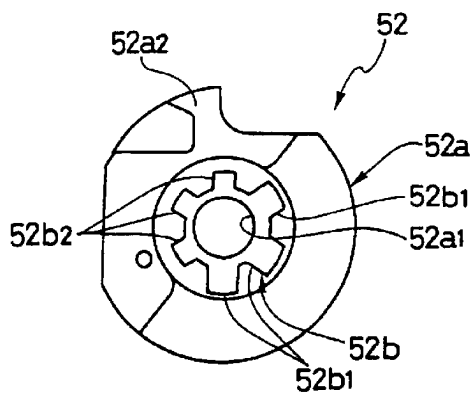


FIG. 13

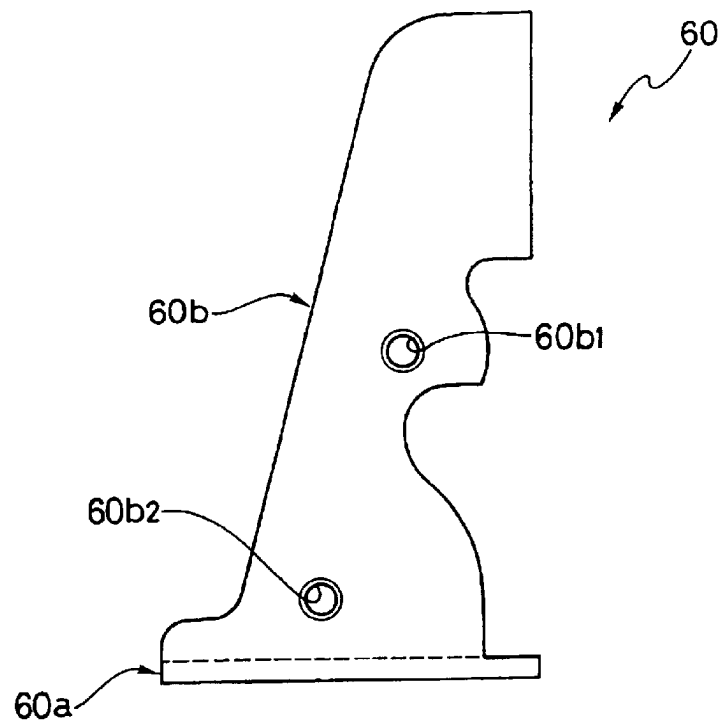


FIG. 15

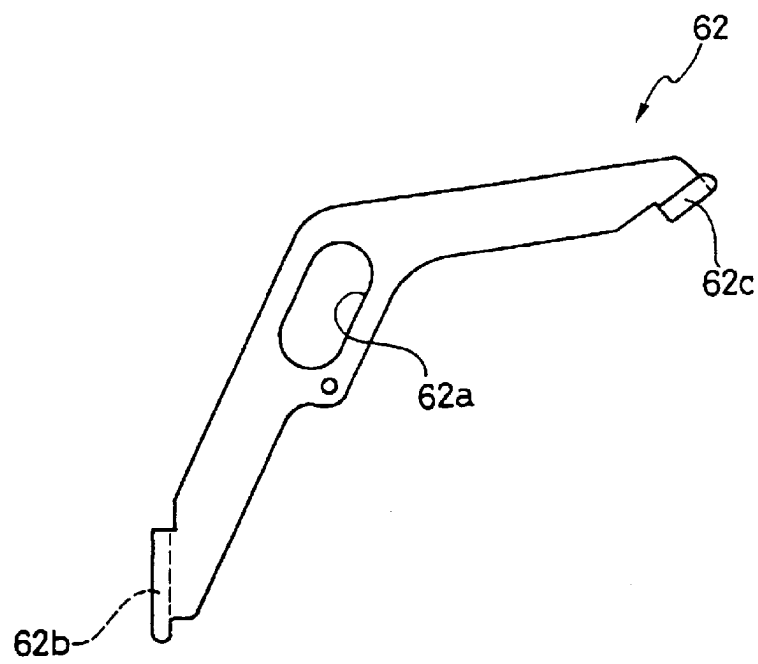


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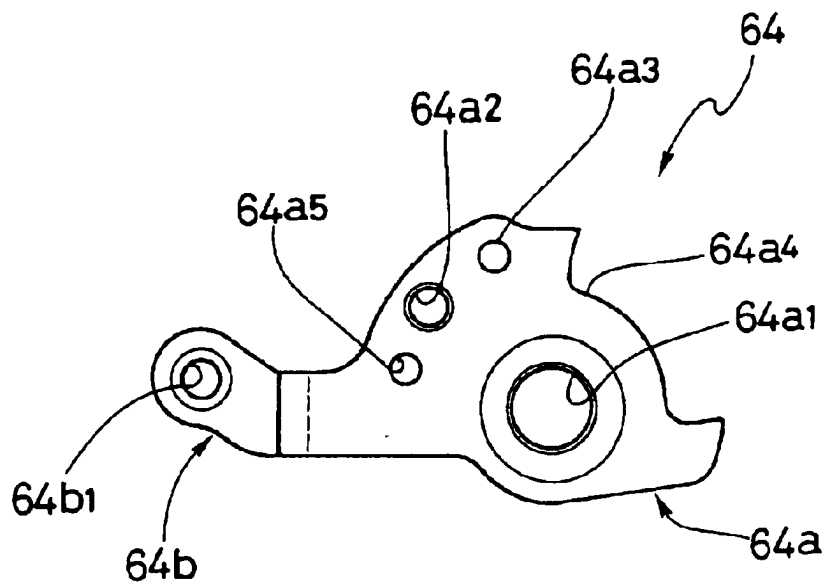


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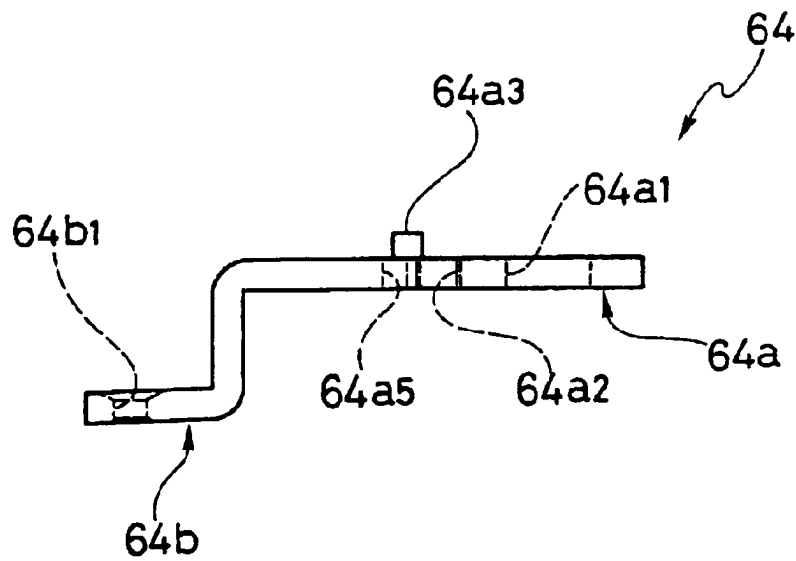


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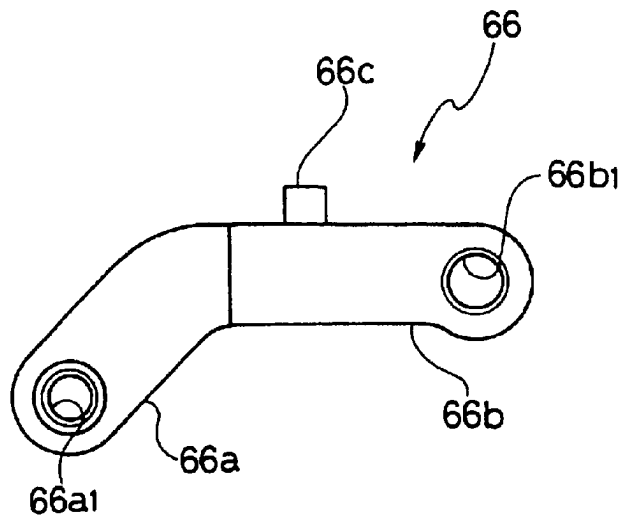


FIG. 19

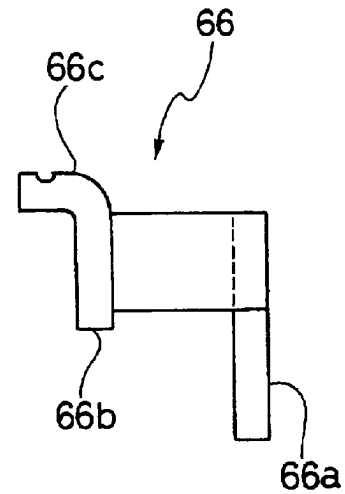


FIG. 20

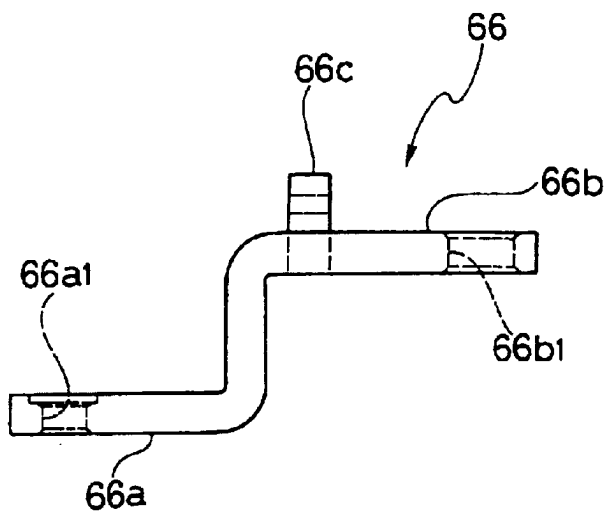


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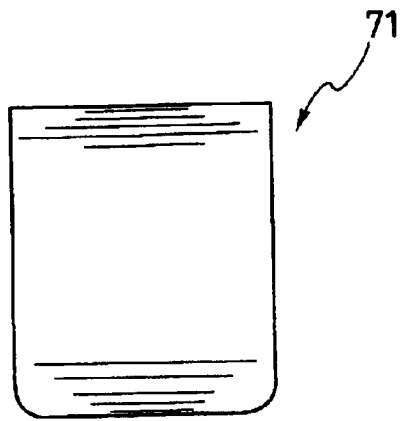


FIG. 22

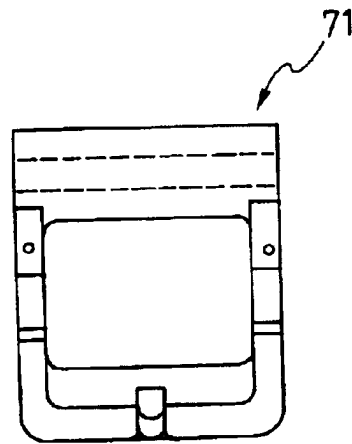


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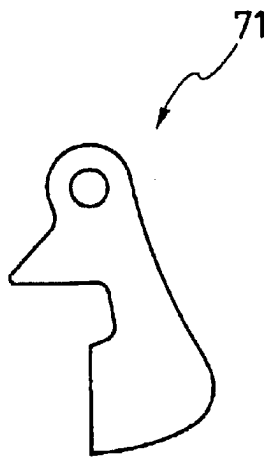


FIG. 24



FIG. 25

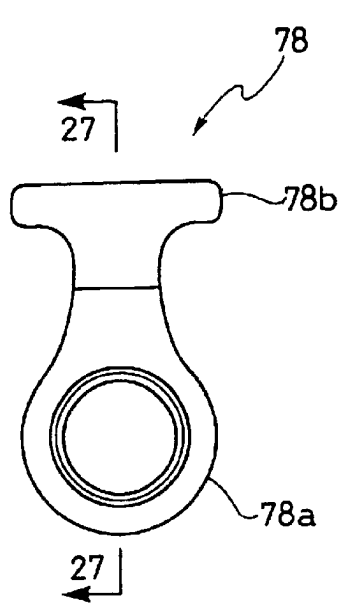


FIG. 26

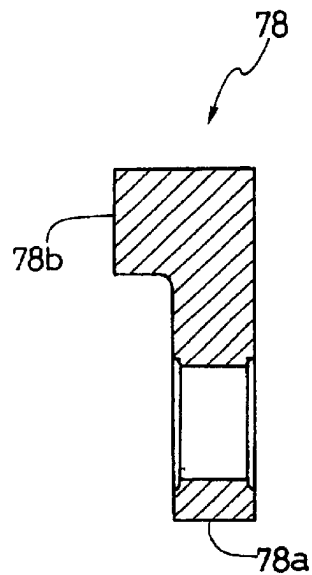


FIG. 27

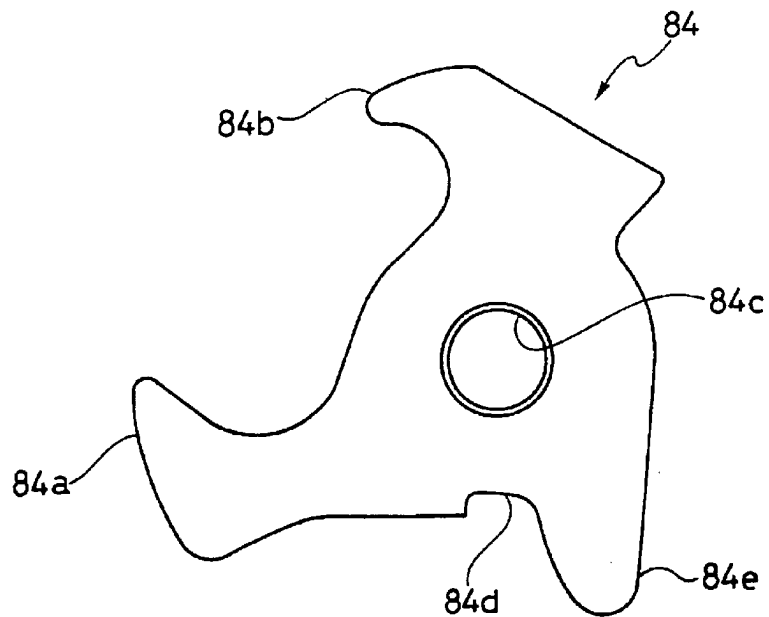


FIG. 28

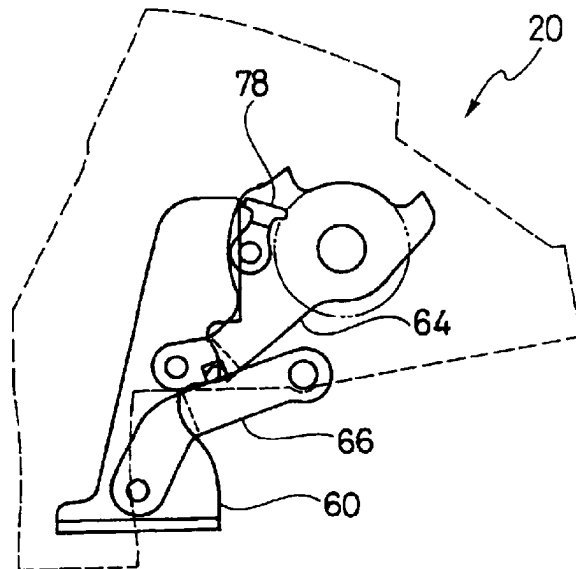


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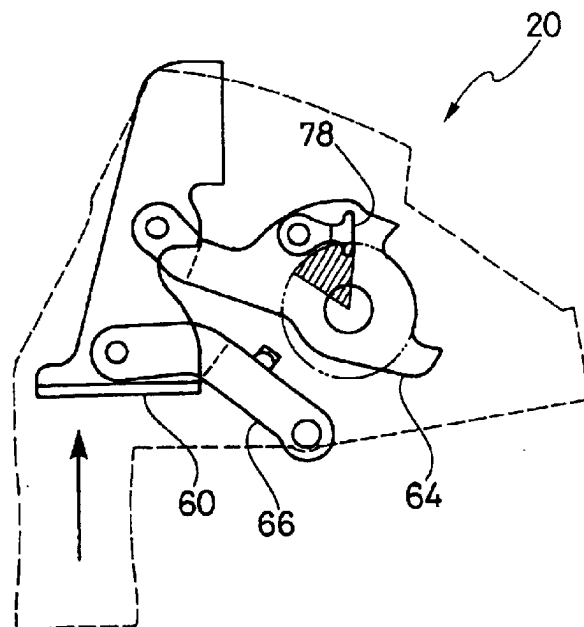


FIG. 30

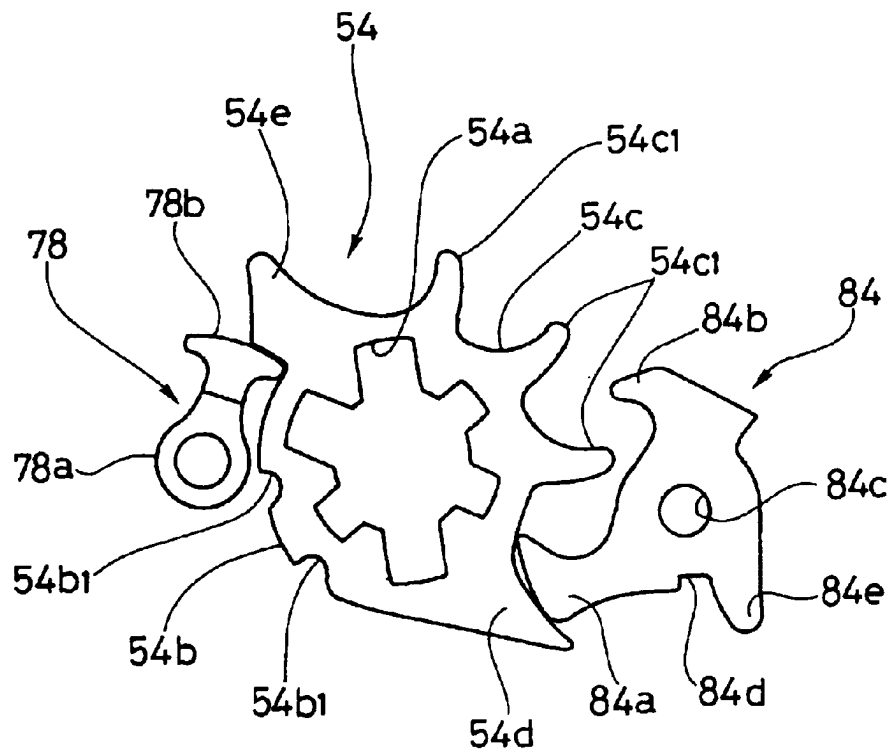


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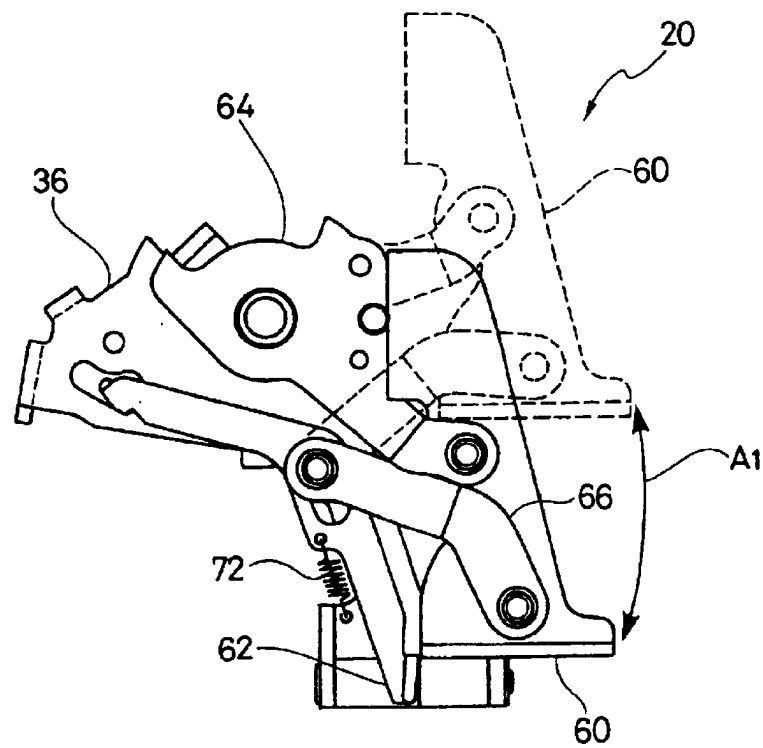


FIG. 32

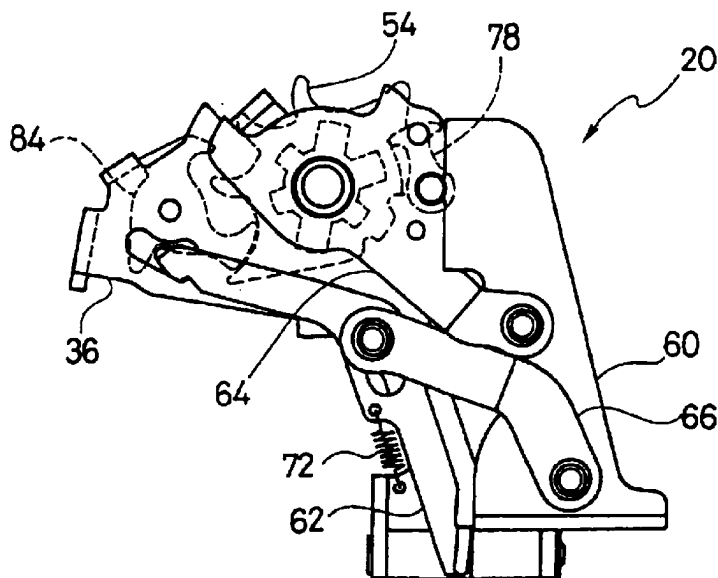


FIG. 33

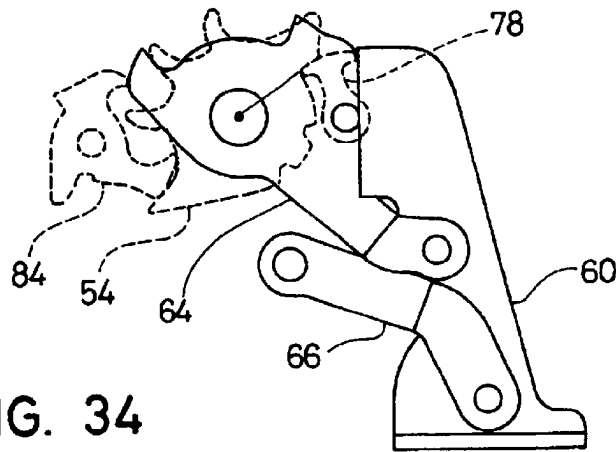


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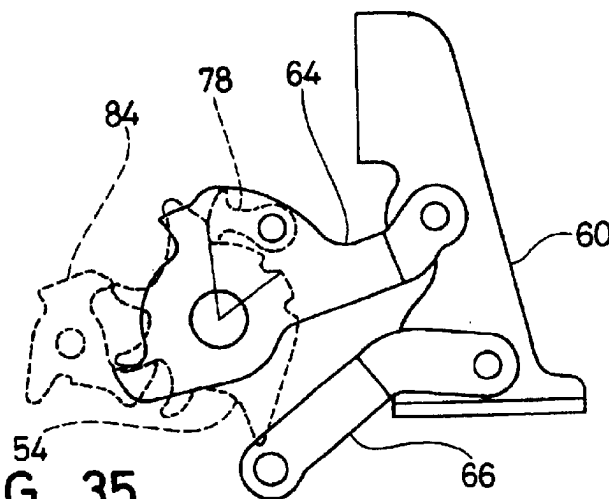


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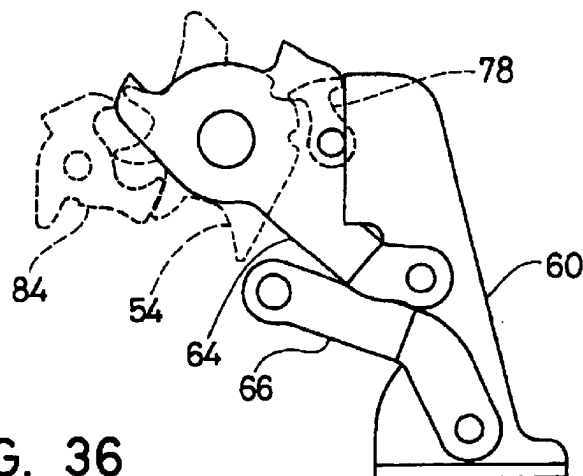


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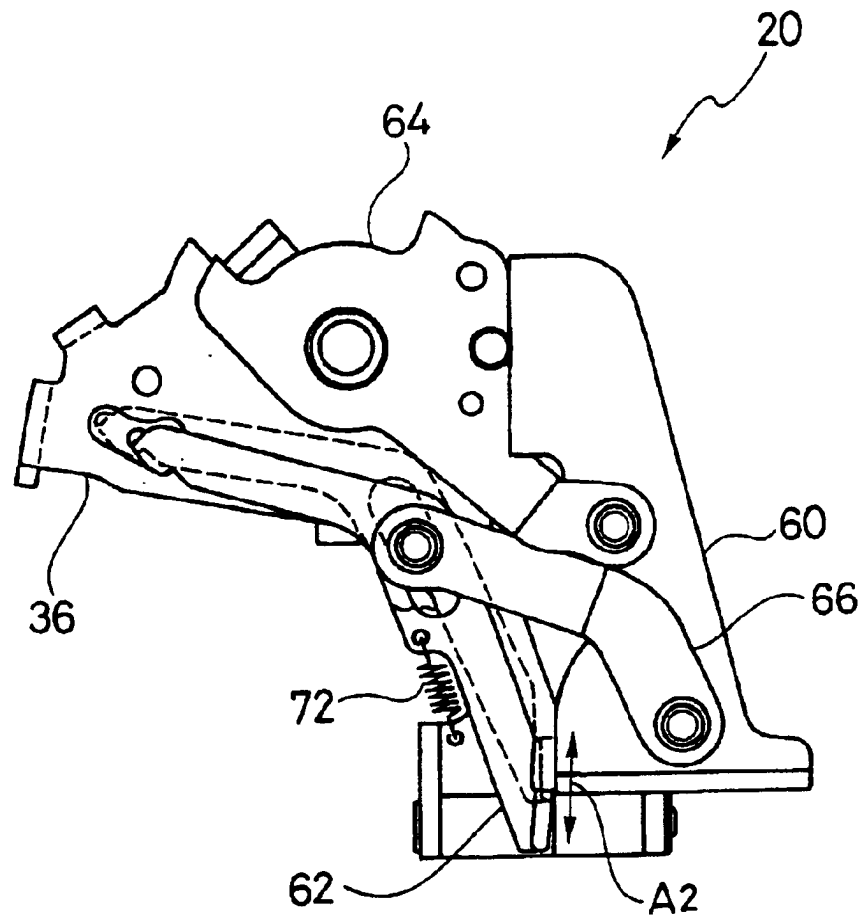
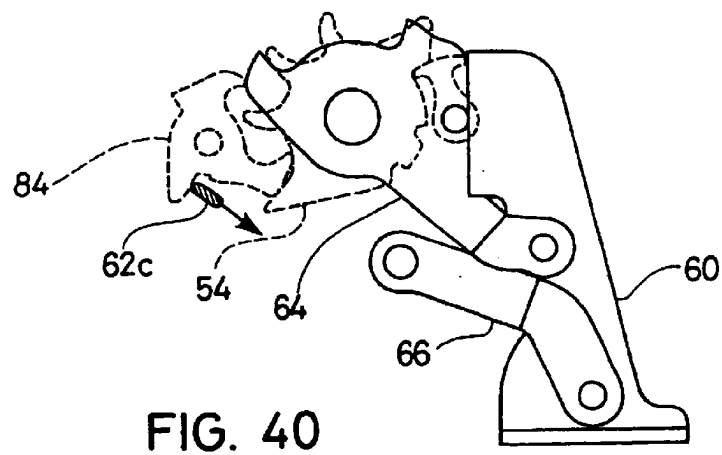
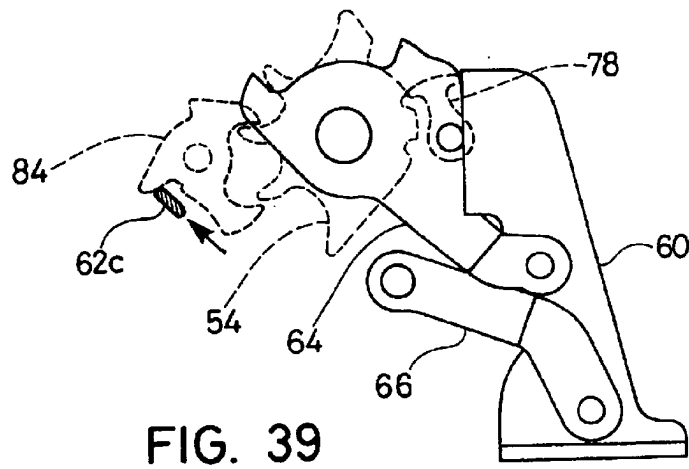
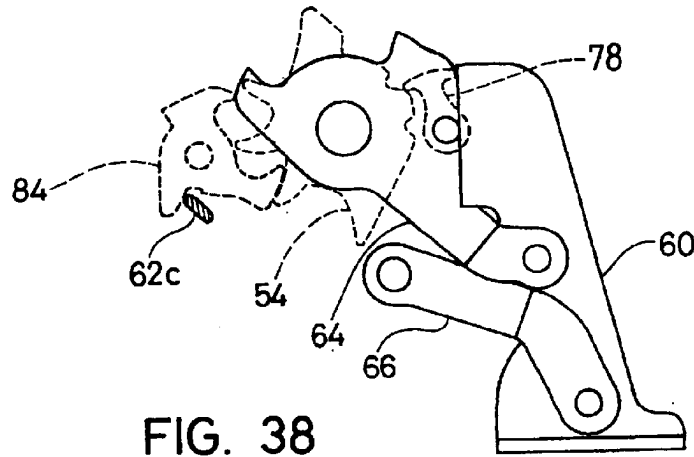
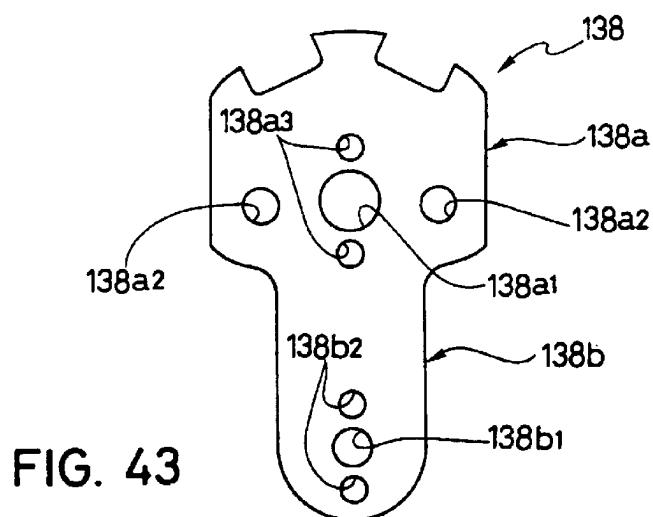
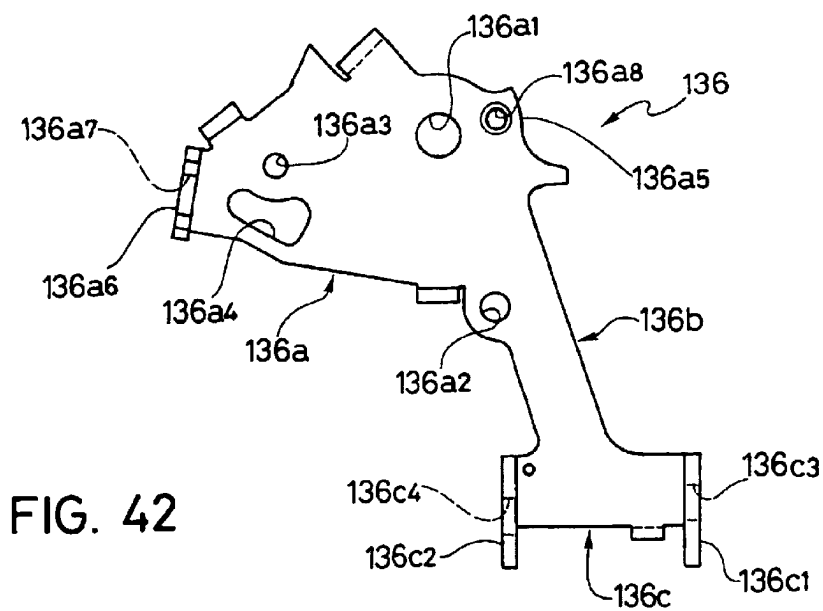
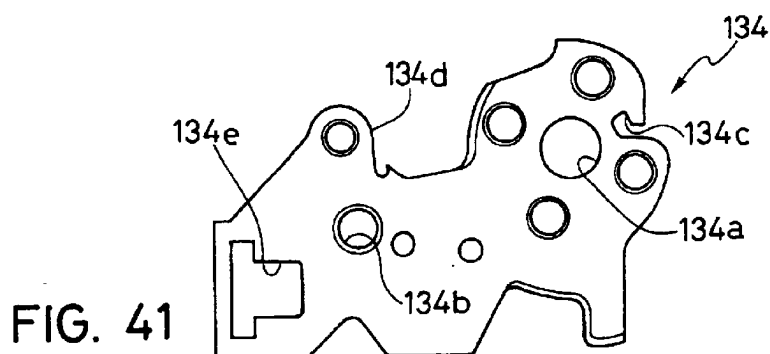


FIG. 37





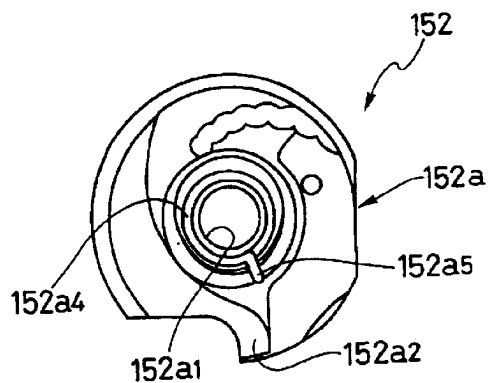


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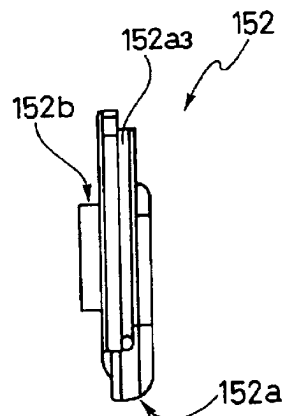


FIG. 45

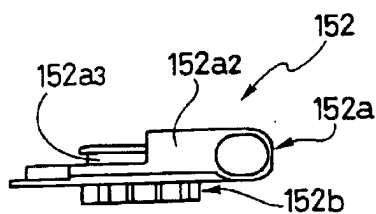


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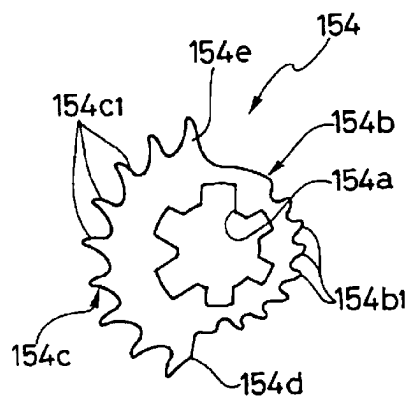


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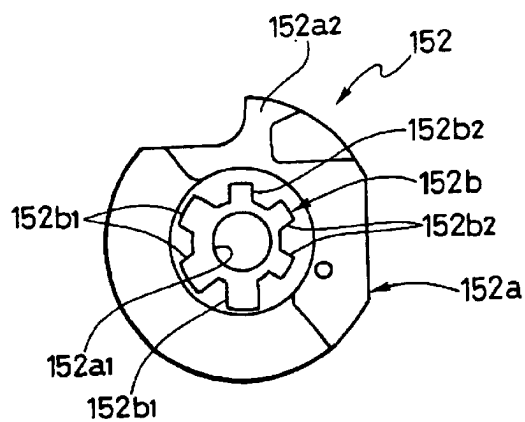


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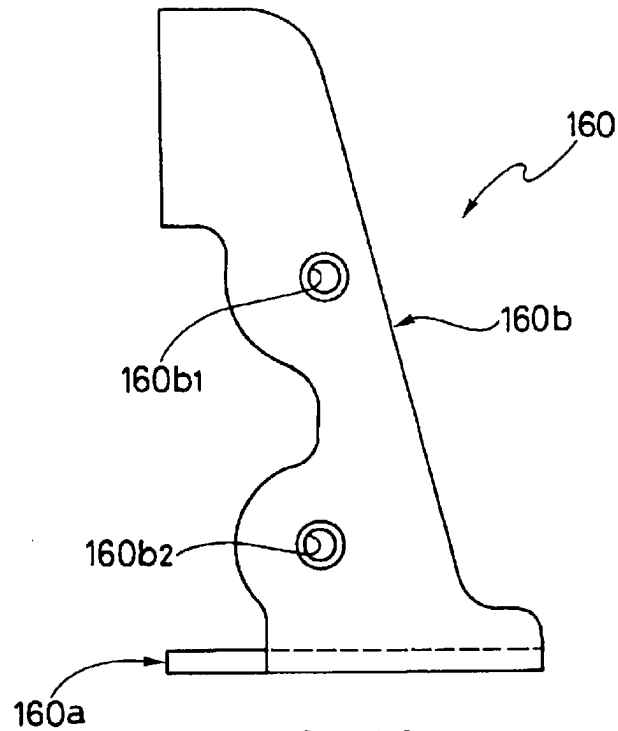


FIG. 49

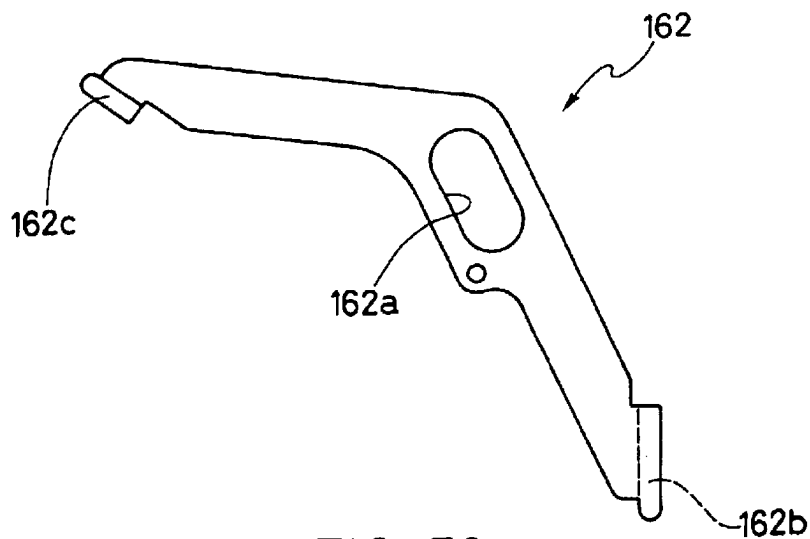


FIG. 50

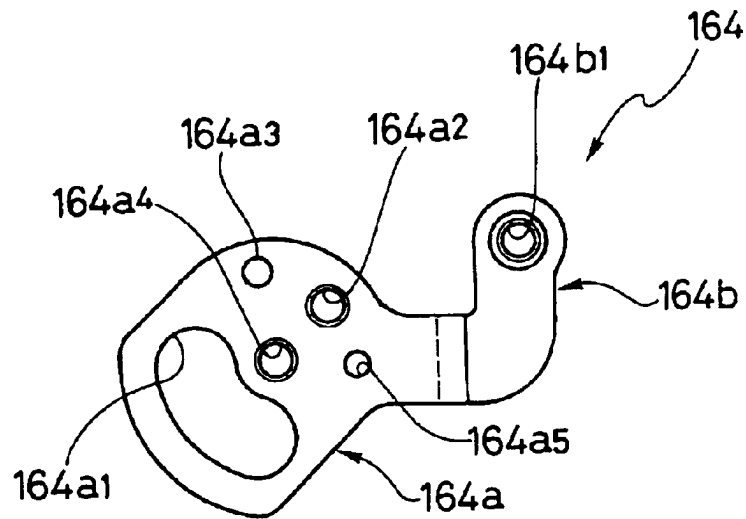


FIG. 51

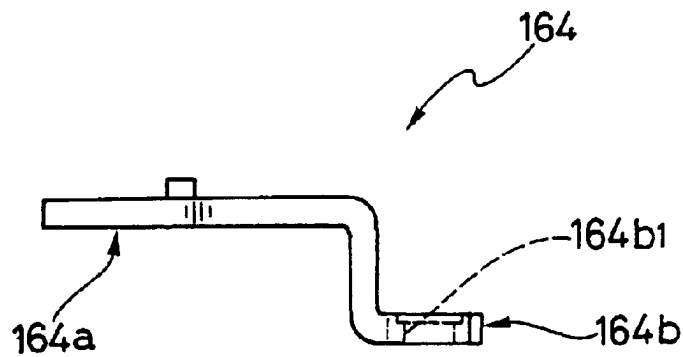


FIG. 52

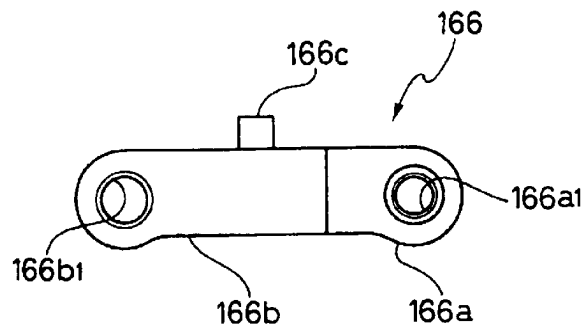


FIG. 53

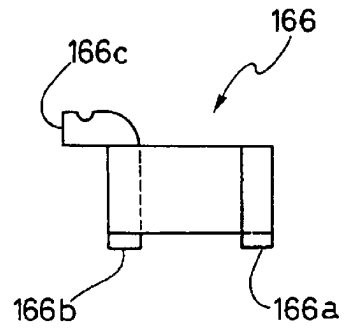


FIG. 54

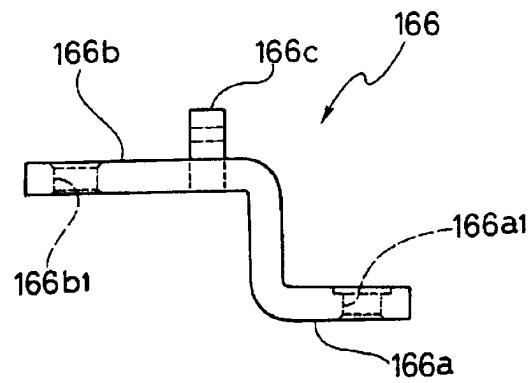


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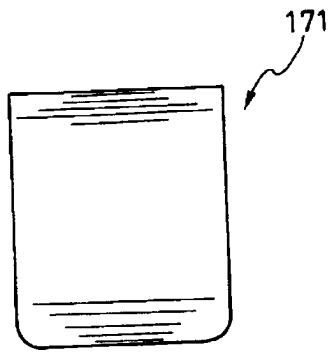


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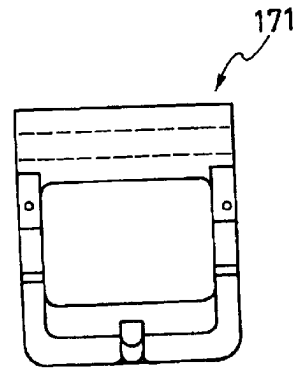


FIG. 57

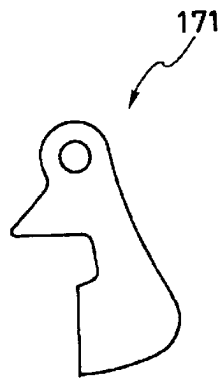


FIG. 58



FIG. 59

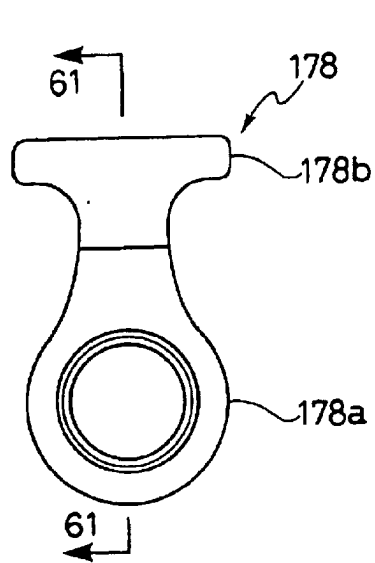


FIG. 60

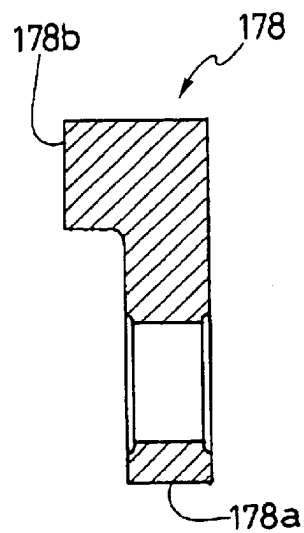


FIG. 61

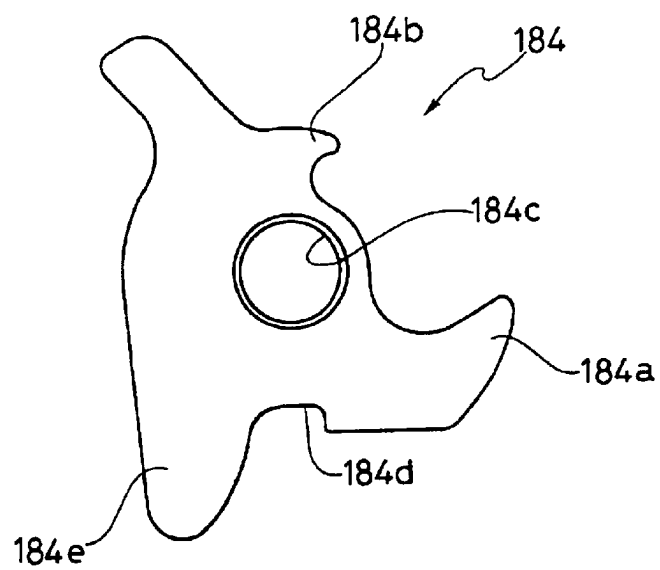


FIG. 62

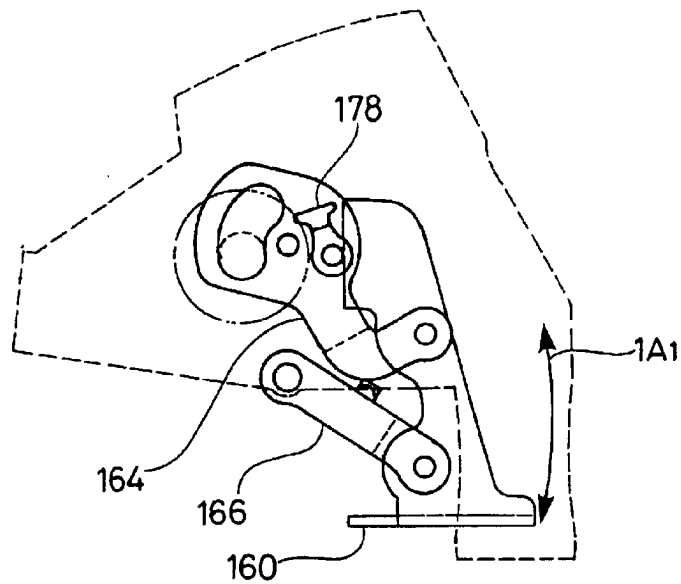


FIG. 63

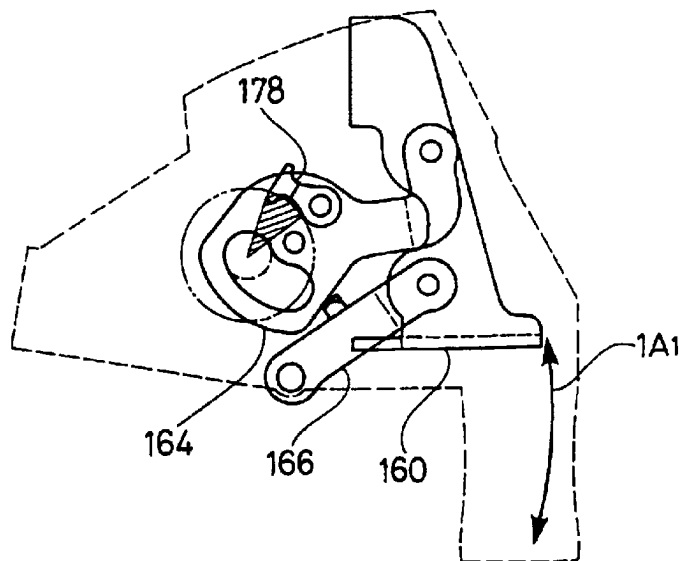


FIG. 64

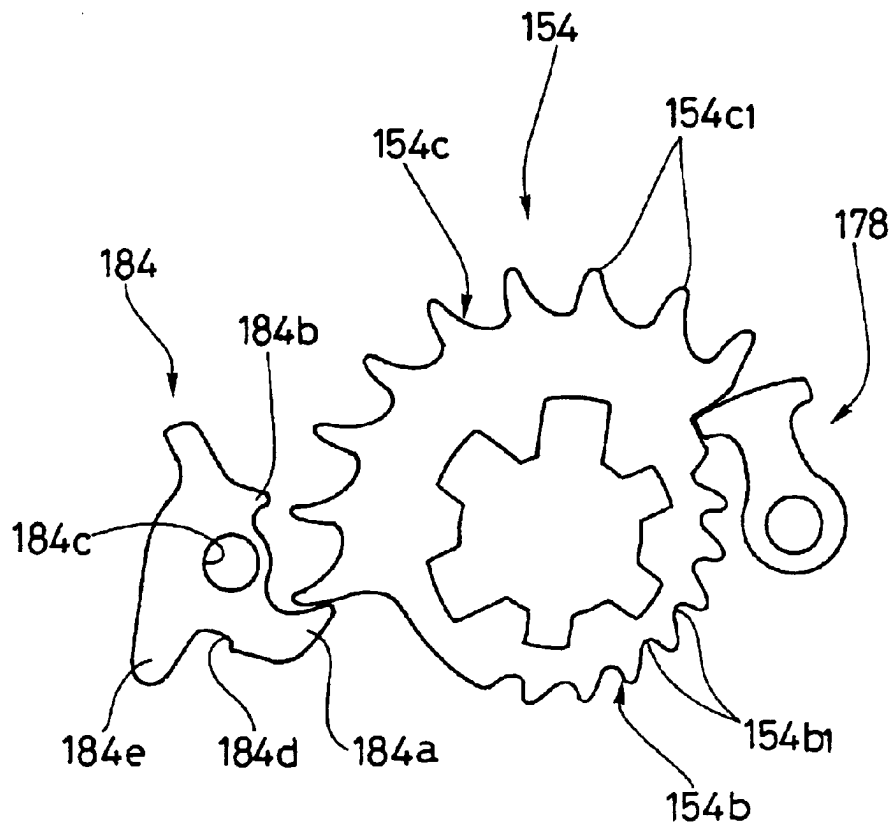
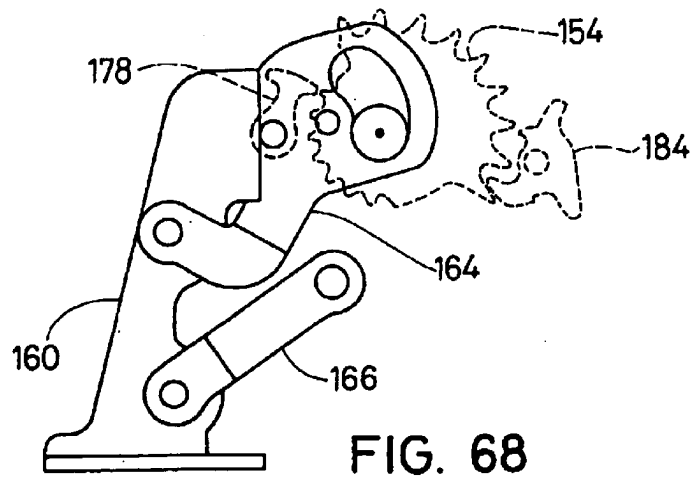
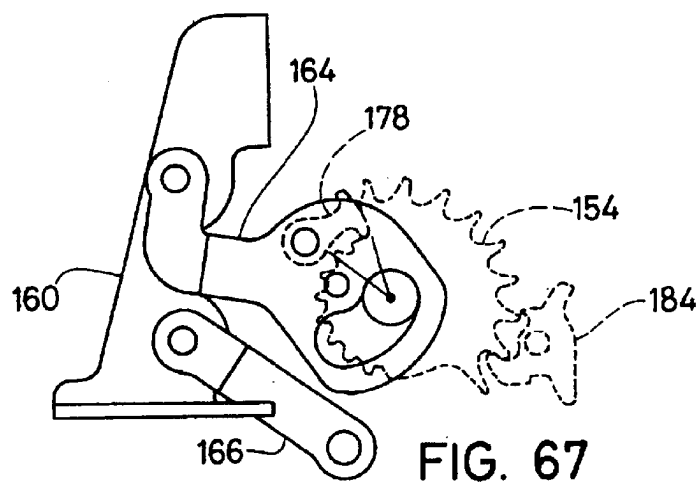
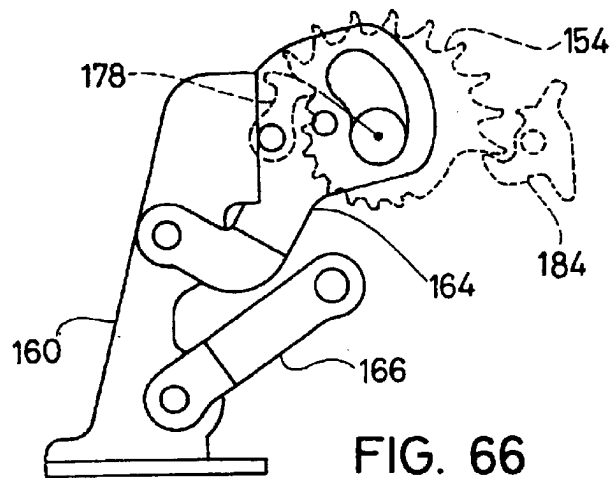


FIG. 65



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BICYCLE SHIFT OPERATING DEVICE FOR BICYCLE TRANSMISSION

CROSS-REFERENCE TO RELATED APPLICATIONS

This application is a divisional application of U.S. patent application Ser. No. 10/041,557 filed on Jan. 10, 2002, now U.S. Pat. No. 6,694,840. The entire disclosure of U.S. patent application Ser. No. 10/041,557, now U.S. Pat. No. 6,694,840, is hereby incorporated herein by reference.

BACKGROUND OF THE INVENTION

1. Field of the Invention

This invention generally relates to a bicycle shift operating device. More specifically, the present invention relates to a shift operating device in which the shifting levers are moved in a substantially parallel direction.

2. Background Information

Bicycling is becoming an increasingly more popular form of recreation as well as a means of transportation. Moreover, bicycling has become a very popular competitive sport for both amateurs and professionals. Whether the bicycle is used for recreation, transportation or competition, the bicycle industry is constantly improving the various components of the bicycle. One part of the bicycle that has been extensively redesigned is the bicycle transmission. Specifically, a bicycle transmission typically includes front and rear shifting mechanisms designed to operate front and rear derailleurs to move the derailleurs laterally over a plurality of sprockets. The sprockets are usually coupled to the front crank and the rear wheel such that a pedaling force from the ride is transferred to the rear wheel via the chain.

In the past, shifting devices have been utilized that include one or more levers that are pivoted to wind an inner wire of a control cable. Alternatively, rotating hand grips have also been utilized to wind the inner wire of the control cable. The wires are coupled to the front and rear derailleurs to shift the chain over the various sprockets. These prior shifting devices can be complicated and expensive to manufacture and assemble. Moreover, these prior shifting devices can require uncomfortable or unnatural movements of the rider's hands to operate. Furthermore, these prior shifting devices are sometimes heavy and/or cumbersome.

In view of the above, there exists a need for a bicycle shift operating device which overcomes the above mentioned problems in the prior art. This invention addresses this need in the prior art as well as other needs, which will become apparent to those skilled in the art from this disclosure.

SUMMARY OF THE INVENTION

One object of the present invention is to provide a bicycle shift operating device with shift levers that move along substantially parallel paths.

Another object of the present invention is to provide a bicycle shift operating device that is relatively simple and inexpensive to manufacture and assemble.

Still another object of the present invention is to provide a bicycle shift operating device that provides reliable shifts of the front and rear derailleurs.

The foregoing objects can basically be attained by providing a bicycle shift operating device comprising a first operating member, a cable winding mechanism, a follower link and an operating link. The cable winding mechanism is mounted on a first pivot axle at a first fixed location and is

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configured to be selectively retained in one of a plurality of shift positions. The follower link has a first end pivotally coupled to the first operating member and a second end mounted on a second pivot axle at a second fixed location.

5 The operating link has a first end pivotally coupled to the first operating member and a second end mounted on the first pivot axle. The operating link is operatively coupled to the cable winding mechanism to rotate the cable winding mechanism about the first pivot axle between the shift positions in response to movement of the first operating member.

The foregoing objects can also basically be attained by providing a bicycle shift operating device comprising a first operating member, a cable winding mechanism, an operating link and a winding pawl. The cable winding mechanism is mounted on a first pivot axle and is configured to be selectively retained in one of a plurality of shift positions. The operating link has a first end coupled to the first operating member and a second end mounted on a second pivot axle. The second pivot axle is spaced from the first pivot axle and is arranged to be parallel to the first pivot axle. The winding pawl is coupled to the operating link and is operatively engaged with the cable winding mechanism. Movement of the operating link about the second pivot axle in a first rotational direction rotates the cable winding mechanism about the first pivot axle between the shift positions in response to movement of the first operating member.

The foregoing objects can also basically be attained by providing a bicycle shift operating device comprising a first operating member, a second operating member, a cable winding mechanism and a retaining mechanism. The first operating member is arranged to reciprocate in a first direction. The second operating member is arranged to move in a second direction that is substantially parallel to the first direction. The cable winding mechanism is operatively coupled to the first operating member to rotate in a first rotational direction and is operatively coupled to the second operating member to rotate in a second rotational direction that is opposite to the first rotational direction. The retaining mechanism is operatively coupled to the cable winding mechanism to selectively hold the cable winding mechanism in one of a plurality of shift positions.

These and other objects, features, aspects and advantages of the present invention will become apparent to those skilled in the art from the following detailed description, which, taken in conjunction with the annexed drawings, discloses a preferred embodiment of the present invention.

BRIEF DESCRIPTION OF THE DRAWINGS

Referring now to the attached drawings which form a part of this original disclosure:

FIG. 1 is a side elevational view of a bicycle with front and rear shift operating devices coupled thereto in accordance with a preferred embodiment of the present invention;

FIG. 2 is an enlarged perspective view of the front shift operating device in accordance with the present invention;

FIG. 3 is an enlarged perspective view of the rear shift operating device in accordance with the present invention;

FIG. 4 is an exploded perspective view of the front shift operating device illustrated in FIG. 2;

FIG. 5 is an exploded perspective view of the rear shift operating device illustrated in FIG. 3;

FIG. 6 is a partial, diagrammatic cross-sectional view of the front shift operating device illustrated in FIGS. 2 and 4;

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FIG. 7 is a top plan view of a base plate of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 8 is a top plan view of an intermediate plate of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 9 is a top plan view of a lever retaining plate of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 10 is a top plan view of a cable winding of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 11 is a side elevational view of the cable winding member illustrated in FIG. 10;

FIG. 12 is an end elevational view of the cable winding member illustrated in FIGS. 10 and 11;

FIG. 13 is a bottom plan view of the cable winding member illustrated in FIGS. 10–12;

FIG. 14 is a top plan view of a ratchet member of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 15 is a top plan view of a fast operating member of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 16 is a top plan view of a second operating member of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 17 is a top plan view of an operating link of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 18 is a side elevational view of the operating link illustrated in FIG. 17;

FIG. 19 is a top plan view of a follower link of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 20 is a side elevational view of the follower link illustrated in FIG. 19;

FIG. 21 is an end elevational view of the follower link illustrated in FIGS. 19 and 20;

FIG. 22 is a front elevational view a tap lever of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 23 is a rear elevational view of the tap lever illustrated in FIG. 22;

FIG. 24 is a side elevational view of the tap lever illustrated in FIGS. 22 and 23;

FIG. 25 is a cross-sectional view of the tap lever illustrated in FIGS. 22–24 as seen along section line 25–25 of FIG. 22;

FIG. 26 is a top plan view of a pawl member of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 27 is a cross-sectional view of the pawl member illustrated in FIG. 26 as seen along section line 27–27 of FIG. 26;

FIG. 28 is a top plan view of a locking member of the front shift operating device illustrated in FIGS. 2, 4 and 6;

FIG. 29 is a partial top plan view of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member in a rest position and portions broken away for the purpose of illustration;

FIG. 30 is a partial top plan view of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member in a shift position after moving the operating link an operating angle and portions broken away for the purpose of illustration;

FIG. 31 is a top plan view of the pawl member, ratchet member and locking member of the front shift operating device illustrated in FIGS. 2, 4 and 6, showing the engagement/relationship between these members;

FIG. 32 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 showing

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the first shift operating member in a normal rest position in solid lines, and in a shift position in broken lines for the purpose of illustration;

FIG. 33 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 showing the first shift operating member in a normal rest position in solid lines with the ratchet member, pawl member and locking member shown in broken lines for the purpose of illustration;

FIG. 34 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member in a normal rest position and with the ratchet member, pawl member and locking member shown in broken lines in order to illustrate operation of the front shift operating device;

FIG. 35 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member in a shift position and with the ratchet member, pawl member and locking member shown in broken lines in shifted positions in order to illustrate operation of the front shift operating device;

FIG. 36 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member moved back to the normal rest position and with the ratchet member, pawl member and locking member shown in broken lines after being shifted in order to illustrate operation of the front shift operating device;

FIG. 37 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 showing the second shift operating member in a normal rest position in solid lines, and in a shift position in broken lines for the purpose of illustration;

FIG. 38 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the second shift operating member in a normal rest position and with the ratchet member, pawl member and locking member shown in broken lines in shifted positions in order to illustrate operation of the front shift operating device;

FIG. 39 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the second shift operating member in a shift position and with the ratchet member, pawl member and locking member shown in broken lines in intermediate positions in order to illustrate operation of the front shift operating device;

FIG. 40 is a bottom plan view of certain parts of the front shift operating device illustrated in FIGS. 2, 4 and 6 with the second shift operating member moved back to the normal rest position and with the ratchet member, pawl member and locking member shown in broken lines after being shifted in order to illustrate operation of the front shift operating device;

FIG. 41 is a top plan view of a base plate of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 42 is a top plan view of an intermediate plate of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 43 is a top plan view of a lever retaining plate of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 44 is a top plan view of a cable winding of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 45 is a side elevational view of the cable winding member illustrated in FIG. 44;

FIG. 46 is an end elevational view of the cable winding member illustrated in FIGS. 44 and 45;

FIG. 47 is a bottom plan view of the cable winding member illustrated in FIGS. 44–46;

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FIG. 48 is a top plan view of a ratchet member of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 49 is a top plan view of a first operating member of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 50 is a top plan view of a second operating member of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 51 is a top plan view of an operating link of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 52 is a side elevational view of the operating link illustrated in FIG. 51;

FIG. 53 is a top plan view of a follower link of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 54 is a side elevational view of the follower link illustrated in FIG. 53;

FIG. 55 is an end elevational view of the follower link illustrated in FIGS. 53 and 54;

FIG. 56 is a front elevational view of a tap lever of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 57 is a rear elevational view of the tap lever illustrated in FIG. 56;

FIG. 58 is a side elevational view of the tap lever illustrated in FIGS. 56 and 57;

FIG. 59 is a cross-sectional view of the tap lever illustrated in FIGS. 56–58 as seen along section line 59–59 of FIG. 56;

FIG. 60 is a top plan view of a pawl member of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 61 is a cross-sectional view of the pawl member illustrated in FIG. 60 as seen along section line 61–61 of FIG. 60;

FIG. 62 is a top plan view of a locking member of the rear shift operating device illustrated in FIGS. 3 and 5;

FIG. 63 is a partial top plan view of the rear shift operating device illustrated in FIGS. 3 and 5 with the first shift operating member in a rest position and portions broken away for the purpose of illustration;

FIG. 64 is a partial top plan view of the rear shift operating device illustrated in FIGS. 3 and 5 with the first shift operating member in a shift position after moving the operating link an operating angle and portions broken away for the purpose of illustration;

FIG. 65 is a top plan view of the pawl member, ratchet member and locking member of the rear shift operating device illustrated in FIGS. 3 and 5, showing the engagement/relationship between these members;

FIG. 66 is a bottom plan view of certain parts of the rear shift operating device illustrated in FIGS. 3 and 5 with the first shift operating member in a normal rest position and with the ratchet member, pawl member and locking member shown in broken lines in order to illustrate operation of the rear shift operating device;

FIG. 67 is a bottom plan view of certain parts of the rear shift operating device illustrated in FIGS. 3 and 5 with the first shift operating member in a shift position and with the ratchet member, pawl member and locking member shown in broken lines in shifted positions in order to illustrate operation of the rear shift operating device; and

FIG. 68 is a bottom plan view of certain parts of the rear shift operating device illustrated in FIGS. 2, 4 and 6 with the first shift operating member moved back to the normal rest position and with the ratchet member, pawl member and locking member shown in broken lines after being shifted in order to illustrate operation of the rear shift operating device.

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DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENTS

Referring initially to FIGS. 1–3, a bicycle 10 is illustrated with a front (first) shift operating device 20 and a rear (second) shift operating device 22 is illustrated in accordance with a preferred embodiment of the present invention. Bicycle 10 basically includes a frame 12, a drive train or transmission 14, a front wheel 16 and a rear wheel 18. The frame 12 includes a handlebar 13 pivotally coupled thereto. Specifically, the handlebar 13 is fixedly coupled to the front fork of the frame 12 to steer the bicycle 10 via the front wheel 16. The drive train or transmission 14 includes a front derailleur 15, a rear derailleur 17, a chain C, a plurality of rear sprockets RS coupled to the rear wheel 18 and a front crank FC with a plurality of front sprockets FS coupled thereto. The front and rear derailleurs 15 and 17 are coupled to the frame 12 to move/shift the chain C laterally between the various sprockets FS and RS in a relatively conventional manner. The front sprockets FS are coupled to the front crank FC, while the rear sprockets RS are coupled to the rear wheel 18 via a free wheel to selectively rotate the rear wheel 18 via the chain C in order to propel the bicycle 10 in a conventional manner.

The front shift operating device 20 is operatively coupled to the front derailleur 15 via a front (first) bowden control cable 21 in order to shift the front derailleur 15 laterally over the front sprockets FS in a relatively conventional manner. Similarly, the rear shift operating device 22 is operatively coupled to the rear derailleur 17 via a rear (second) bowden control cable 23 to shift the rear derailleur 17 laterally over the rear sprockets RS in a relatively conventional manner. The front and rear control cables 21 and 23 are identical except for their lengths, and are basically conventional. In other words, each of the control cables 21 and 23 basically includes an inner wire slidably received within an outer casing.

A front brake 19a is coupled to the front fork of the bicycle frame 12, while a rear brake 19b is coupled to the rear triangle of the bicycle frame 12. The front and rear brakes 19a and 19b selectively apply a friction force to the front and rear rims of the front and rear wheels 16 and 18 to selectively stop the rotation of the front and rear wheels 16 and 18, respectively in a conventional manner.

The various parts of the bicycle 10 are conventional, except for the front and rear shift operating devices 20 and 22 as discussed below. Thus, the remaining parts of the bicycle 10 will not be discussed or illustrated in detail herein, except as they relate to the front and rear shift operating devices 20 and 22. Accordingly, it will be apparent to those skilled in the art from this disclosure that various modifications can be made to the various components or parts of the bicycle 10 without departing from the scope of the present invention.

As mentioned above, the front shift operating device 20 is operatively coupled to the front derailleur 15 via the front control cable 21, while the rear shift operating device 22 is operatively coupled to the rear derailleur 17 via the rear control cable 23. In the illustrated embodiment, each of the front and rear shift operating devices 20 and 22 includes an integrated brake operating device, as discussed below in more detail. However, it will be apparent to those skilled in the art from this disclosure that the shift operating devices 20 and 22 could be designed to be separate from the brake operating devices.

Front Shift Operating Device

Referring now to FIGS. 1, 2, 4 and 6, the front shift operating device 20 will now be discussed in more detail.

The front shift operating device **20** basically includes a mounting assembly **24**, a winding mechanism **26**, an operating mechanism **28** and a retaining mechanism **30** coupled together to shift the front derailleur **15** and the chain **C** of the drive train or transmission **14** between the front sprockets **FS**. In the illustrated embodiment, the front crank **FC** preferably has at least three front sprockets **FS** coupled thereto. In any event, the front shift operating device **20** preferably has a number of shift positions that corresponds to the number of shift positions of the front derailleur **15**. Of course, it will be apparent to those skilled in the art from this disclosure that the front derailleur **15** and the front shift operating device **20** could be designed with a different number of shift positions if needed and/or desired.

The winding mechanism **26**, the operating mechanism **28** and the retaining mechanism **30** are coupled to the mounting assembly **24**. The winding mechanism **26** is controlled by the operating mechanism **28** and the retaining mechanism **30** to selectively maintain the front derailleur **15** in one of a plurality of shift positions via the control cable **21**. More specifically, the operating mechanism **28** and the retaining mechanism **30** control rotation of the winding mechanism **26** to selectively retain the winding mechanism **26** in one of a plurality of shift positions. Thus, the inner wire of the control cable **21** and the front derailleur **15** are also selectively retained in one of a plurality of shift positions. The inner wire of the front control cable **21** is coupled to the winding mechanism **26** such that rotation of the winding mechanism **26** takes-up or lets-out the inner wire of the control cable **21** to actuate/move/shift the front derailleur **15** between the front sprockets **FS**.

The mounting assembly **24** basically includes a plurality of fixed members coupled together to form a shift operating device housing that supports the winding mechanism **26**, the operating mechanism **28** and the retaining mechanism **30**. More specifically, the mounting assembly **24** basically includes a main mounting portion **32**, a base plate **34**, an intermediate plate **36** and a lever retaining plate **38**. The base plate **34**, the intermediate plate **36** and the lever retaining plate **38** are basically fixedly coupled to the main mounting portion **32** by a main fixing bolt **40** (first pivot axle) and a secondary fixing bolt **42** (second pivot axle) to form the shift operating device housing. Various other parts of the front shift operating device **20** (i.e., parts of the winding mechanism **26**, operating mechanism **28** and retaining mechanism **30**) are either movably or non-movably coupled to the mounting assembly **24**, as discussed below in more detail.

The main mounting portion **32** basically includes a main/shift mounting plate **32a**, a handlebar mounting bracket **32b** and a brake lever mounting bracket **32c** as best seen in FIG. 4. A brake lever **33** is pivotally coupled to the brake lever mounting bracket **32c** of the main mounting portion **32** in the illustrated embodiment. The brake lever **33** is operatively coupled to the front brake **19a** by a bowden brake cable in a conventional manner. Preferably, the main/shift mounting plate **32a**, the handlebar mounting bracket **32b** and the brake lever mounting bracket **32c** are integrally formed together as a one-piece unitary member that is coupled to the handlebar **13** via the mounting bracket **32b**. Thus, an integral front shift/brake operating device **20** is preferably provided. The main mounting portion **32** is preferably constructed of a lightweight rigid material such as cast aluminum. Of course, any suitable material could be utilized if needed and/or desired.

The base plate **34**, the intermediate plate **36** and the retaining plate **38** are preferably fixedly coupled to the main/shift mounting plate **32a** via the fixing bolts **40** and **42**

in a spaced arrangement to accommodate parts of the winding mechanism **26**, operating mechanism **28** and retaining mechanism **30** therebetween. More specifically, the main/shift mounting plate **32a** includes a main through hole **32a₁** and a blind threaded bore **32a₂** configured to mount the fixing bolts **40** and **42** therein, respectively. A cover **44** and indicating mechanism **46** are also preferably fixedly coupled to the main/shift mounting plate **32a**. However, the cover **44** and the indicating mechanism **46** are relatively conventional. Thus, the cover **44** and the indicating mechanism **46** will not be discussed and/or illustrated in detail herein. Moreover, the cover **44** and the indicating mechanism **46** are omitted from some Figures for the purpose of clarity. In any event, the indicating mechanism **46** is preferably operatively coupled with the shift operating device **20** to indicate the various shift positions in a conventional manner.

As seen in FIGS. 4, 6 and 7, the base plate **34** is a relatively thin, fiat member. The base plate **34** is preferably constructed of a lightweight rigid material such as sheet metal. The base plate **34** is located between the main mounting portion **32** and the intermediate plate **36** and basically has a main through bore **34a**, a positioning mounting bore **34b**, a spring mounting slot **34c**, a projecting section **34d** and a T-shaped opening **34e**. The main through bore **34a** is configured to receive the main fixing bolt **40** therethrough while the mounting bore **34b** is configured to receive part of the retaining mechanism **30**. Specifically, the retaining mechanism **30** is coupled to the positioning mounting bore **34b** and the intermediate plate **36** to prevent rotation of the base plate **34** relative to the main mounting portion **32** as discussed below. Also, a portion of the intermediate plate **36** extends partially through the T-shaped opening **34e** to prevent rotation of the base plate **34**, and acts as a spacer as also discussed below. The spring mounting slot **34c** is configured to engage a portion of the winding mechanism **26** as discussed below. The projecting section **34d** is configured to engage a portion of the retaining mechanism **30** as also discussed below.

As seen in FIGS. 4, 6 and 8, the intermediate plate **36** is preferably formed of several thin flat portions integrally formed together as a one-piece unitary member. Moreover, the intermediate plate **36** is preferably constructed of a lightweight rigid material such as deformed sheet metal. The intermediate plate **36** is located between the base plate **34** and the lever retaining plate **38**, and basically includes a base portion **36a**, a connecting portion **36b**, and a lever mounting portion **36c**. The base portion **36a** is fixedly and non-rotatably coupled to the main/shift mounting plate **32a**. The connecting portion **36b** extends outwardly from the base portion **36a**. The lever mounting portion **36c** is coupled to the free end of the connecting portion **36b**.

The base portion **36a** basically includes a main through hole **36a₁**, a secondary hole **36a₂**, a positioning hole **36a₃**, a guide opening **36a₄** and a release surface **36a₅**. A flange **36a₆** with a mounting hole **36a₇** preferably extends from the base portion **36a** to form a right angle. The main hole **36a₁** is configured to receive the main fixing bolt **40** therethrough, while the secondary hole **36a₂** is configured to receive the secondary fixing bolt **42** therethrough. Thus, the base portion **36a** is non-rotatably coupled to the main mounting portion **32**. The retaining mechanism **30** is coupled to the positioning hole **36a₃** and the hole **34b** such that the base plate **34** is also non-rotatably coupled to the main mounting portion **32** via the base portion **36a**, as discussed below.

The guide opening **36a₄** is configured to receive a portion of the operating mechanism **28** as discussed below in more detail. The release surface **36a₅** is configured to selectively

engage a portion of the operating mechanism 28 as also explained below. The flange 36a₆ extends partially through the T-shaped opening 34e of the base plate 34 to non-rotatably couple the base plate 34 to the intermediate plate 36. The flange 36a₆ also acts as a spacer between the base plate 34 and the intermediate plate 36 due to the stepped configuration of the flange 36a₆. The mounting hole 36a₇ receives a fastener 45 to couple the control cable 21 thereto.

The lever mounting portion 36c includes first and second mounting flanges 36c₁ and 36c₂, respectively. The mounting flanges 36c₁ and 36c₂ include holes 36c₃ and 36c₄, respectively, that are configured to receive a portion of the operating mechanism 28, as also discussed below in more detail. The connecting portion 36b extends between the lever mounting portion 36c and the base portion 36a such that the lever mounting portion 36c is spaced outwardly from the base portion 36a.

As seen in FIGS. 4, 6 and 9, the lever retaining plate 38 is preferably a thin flat member. The lever retaining plate 38 is preferably constructed of a lightweight rigid material such as sheet metal. The lever retaining plate 38 is located on an opposite side of the intermediate plate 36 from the base plate 34. Thus, the lever retaining plate 38 is located furthest from the main mounting portion 32. The lever retaining plate 38 is fixedly and non-rotatably coupled to the main/shift plate 32a via the fixing bolts 40 and 42. The lever retaining plate 38 basically includes a main support portion 38a and an extending portion 38b.

The main support portion 38a has a main hole 38a₁, a pair of opposed auxiliary holes 38a₂, and a pair of opposed retaining holes 38a₃. The main hole 38a₁ is configured to receive the main fixing bolt 40 therethrough. In this embodiment, the auxiliary holes 38a₂ are not used. However, one of these auxiliary holes 38a₂ is used in the rear shift operating device 22, discussed below. The pair of retaining holes 38a₃ are configured to receive projections of a main nut plate 41 to prevent loosening of the main fixing bolt 40 after the parts of the mounting assembly 24 are coupled together.

The extending portion 38b includes a secondary hole 38b₁ and a pair of retaining holes 38b₂. The secondary hole 38b₁ is configured to receive the secondary fixing bolt 42 therethrough. The retaining holes 38b₂ are configured to receive projections of a secondary bolt plate 43 to prevent loosening of the secondary fixing bolt 42 after the parts of the mounting assembly 24 are coupled together. Thus, the lever retaining plate 38, the intermediate plate 36, the base plate 34 and the main/shift plate 32a are fixedly and non-rotatably coupled together to form the housing or mounting assembly 24.

Referring to FIGS. 4, 6 and 10-14, the winding mechanism 26 will now be discussed in more detail. The winding mechanism 26 is mounted on the main fixing bolt 40 between the base plate 34 and the intermediate plate 36 of the mounting assembly 24. The winding mechanism 26 basically includes a cable winding member 52, a ratchet member 54, a main biasing member 56 and a unit spacer 58. In the illustrated embodiment, the winding member 52 and the ratchet member 54 are preferably separate members that are non-rotatably coupled together. The main biasing member 56 is preferably a torsion spring that normally biases the winding member 52 and the ratchet member 54 in a predetermined rotational direction, e.g., the counter-clockwise direction CCW as seen in FIG. 4. The winding mechanism 26 is operatively coupled to the operating mechanism 28 and the retaining mechanism 30 to selectively retain the winding member 52 in one of a plurality of shift positions.

The winding member 52 is preferably a step-shaped member integrally formed as a one-piece unitary member. Moreover, the winding member 52 is preferably constructed of a lightweight rigid material such as hard plastic. The winding member basically includes a winding portion 52a and a projecting portion 52b. A central through opening 52a₁ extends through both the winding portion 52a and the projecting portion 52b and is configured to receive the main fixing bolt 40 therethrough. Thus, the winding member 52 is coupled to the mounting assembly 24 for rotation about the main fixing bolt 40. The ratchet member 54 is preferably non-rotatably coupled to the winding member 52 and is located between the winding member 52 and the intermediate plate 36. Thus, the ratchet member 54 rotates with the winding member 52. As explained below, the rotational movement of the winding member 52 is controlled by the ratchet member 52.

The winding portion 52a basically has a cable mounting section 52a₂, a winding surface 52a₃ and a spring receiving recess 52a₄. The cable mounting section 52a₂ includes a through opening configured to receive the inner wire of the control cable 21 and a recess configured to receive a nipple or abutment of the inner wire of the control cable 21 in a conventional manner. The winding surface 52a₃ is a curved groove-shaped surface which extends circumferentially away from the cable mounting section 52a₂. The spring receiving recess 52a₄ is a substantially annular recess with a larger diameter than the central through opening 52a₁. Moreover, the spring receiving recess 52a₄ includes a cutout or notch 52a₅ configured to receive one end of the spring 56 as discussed below. The spring receiving recess 52a₄ preferably has a radial width sufficient to receive a portion of the main spring 56 as also discussed below in more detail. The winding portion 52a also preferably includes a stepped groove 52a₆ configured to receive a portion (not shown) of the indicating mechanism 46.

The projecting portion 52b basically includes a plurality of large splines 52b₁ and a plurality of small splines 52b₂ that are configured to non-rotatably engage the ratchet member 54. The large splines 52b₁ are arranged on an opposite side of the central through opening 52a₁ from the small splines 52b₂. Moreover, the splines 52b₁ and 52b₂ are oriented such that the ratchet member 54 is arranged in a certain configuration in order to properly engage both the operating mechanism 28 and the retaining mechanism 30, as discussed below.

The ratchet member 54 is preferably a thin flat member. Preferably, the ratchet member 54 is constructed of a lightweight rigid material such as sheet metal. The ratchet member 54 basically includes a splined central opening 54a, an outer operating surface 54b and an outer positioning surface 54c. The splined opening 54a is configured to receive the projecting portion 52b of the winding member 52 to non-rotatably couple the ratchet member 54 to the winding member 52 in a predetermined orientation. In other words, the opening 54a mates with the large splines 52b₁ and the small splines 52b₂ of the projecting portion 52b so that the winding member 52 rotates with the ratchet member 54 as if they were a single piece.

The operating surface 54b is configured to engage a portion of the operating mechanism 28, which is discussed below. More specifically, the operating surface 54b includes a plurality (three) of grooves 54b₁ designed to selectively engage a portion of the operating mechanism 28. The positioning surface 54c is configured to engage a portion of the retaining mechanism 30 (which is also discussed below) to selectively hold and retain the ratchet member 54 and the

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winding member 52 in one of the plurality of shift positions. More specifically, the positioning surface 54c includes a plurality (three) of projections or teeth 54c₁ which engage a portion of the retaining mechanism 30. The ratchet member 54 also preferably includes a pair of stops 54d and 54e. Specifically, the stop 54d is a rest stop while the stop 54e is a fully shifted stop. The stops 54d and 54e selectively engage portions of the retaining mechanism 30 to control movements of the winding member 52 and the ratchet member 54.

As best seen in FIG. 4, the biasing member 56 is preferably a torsion spring that is arranged to apply an urging force on the winding member 52 and the ratchet member 54 to rotate about the main fixing bolt 40. The biasing member (spring) 56 basically includes a first end 56a, a second end 56b and a coiled portion 56c extending between the first and second ends 56a and 56b. The first end 56a is arranged in the spring mounting slot 34c of the base plate 34, while the second end 56b is arranged in the notch or cutout 52a₅ extending from the spring receiving recess 52a₄ of the winding member 52. The coiled portion 56c is partially received in the spring receiving recess 52a₄. The unit spacer 58 is disposed in the central through opening 52a₁ of the winding member 52.

The unit spacer 58 is preferably a tubular member constructed of lightweight rigid material with one end contacting the lower surface of the base plate 34 and the other end contacting the intermediate plate 36. The spring 56 is configured to normally bias the winding member 52 in a counter-clockwise direction CCW as viewed from above in FIG. 4. Thus, the operating mechanism 28 moves the winding member 52 via the ratchet member 54 in a clockwise direction against the urging force of the main biasing member 56. More specifically, the operating mechanism 28 preferably moves the winding member 52 to one of three predetermined shift positions, as discussed below. The retaining mechanism 30 selectively retains the winding member 52 and the ratchet member 54 in one of the three shift positions, as also discussed below.

Referring now to FIGS. 4, 6 and 15–27, the operating mechanism 28 will now be discussed in more detail. The operating mechanism 28 basically includes a first operating member 60, a second operating member 62, an operating link 64 and a follower link 66. The first operating member 60 together with the operating link 64, the follower link 66 and the mounting assembly 24 form a four-bar linkage that controls movement of the first operating member 60. More specifically, the first operating member 60 is pivotally coupled to both the operating link 64 and the follower link 66 to move along a first arc A₁. The operating link 64 is pivotally coupled to the main fixing bolt 40, while the follower link 66 is pivotally coupled to the secondary fixing bolt 42. The first arc A₁ has a relatively flat curvature such that the first operating member 60 moves substantially in a straight line as best seen in FIG. 32.

The operating link 64 preferably has a pawl mechanism 68 operatively coupled thereto. The pawl mechanism 68 is configured to engage the operating surface 54b of the ratchet member 54 when the first operating member 60 is pushed or moved by the rider from the normal rest position to the shift position. Thus, when the first operating member 60 is moved in a substantially linear direction along the arc A₁, the operating link 64 is also moved such that the pawl mechanism 68 rotates the ratchet member 54 one shift position against the urging force of the biasing member 56. Thus, the winding member 52 is also rotated to pull the inner wire of the shift cable 21 and shift the front derailleur 15 to one of its shift positions.

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When the first operating member 60 moves back to the normal rest position, the pawl mechanism 68 engages the release surface 36a₅ of the intermediate plate 36 to disengage the pawl mechanism from the ratchet member 54. However, the retaining mechanism 30 engages the ratchet member 54 to hold the winding member 52 and the ratchet member 54 in the new shift position against the urging force of the spring 56. In the illustrated embodiment, movement of the first operating member 60 causes only one shift of the winding member 52 from one shift position to the next adjacent shift position in one rotational direction.

The second operating member 62 is operatively coupled to the retaining mechanism 30 to selectively release the ratchet member 54 and the winding member 52. In other words, the rider pushes or moves the second operating member 62 to disengage the retaining mechanism 30 from the ratchet member 54, which allows the winding member 52 and the ratchet member 54 to rotate under the urging force of the spring 56. The retaining mechanism 30 is configured and arranged so that the winding member 52 and the ratchet member 54 rotate only one shift position for each push of the second operating member 62. In other words, the ratchet member 54 engages the retaining mechanism 30 after rotating one shift position. Thus, in the illustrated embodiment, movement of the second operating member 62 causes only one shift of the winding member 52 from one shift position to the next adjacent shift position in another rotational direction opposite to the rotational direction caused by moving the first operating member 60.

As best seen in FIGS. 4 and 15, the first operating member 60 is a thin flat member. The first operating member 60 is preferably constructed of lightweight rigid material such as deformed sheet metal. The first operating member 60 basically includes an actuating portion 60a and a link portion 60b. The link portion 60b is preferably perpendicular to the actuating portion 60a. The link portion 60b includes a pair of holes 60b₁ and 60b₂ that are configured to receive the upper ends of a pair of pivot pins 69 and 70, respectively. The pivot pin 69 is pivotally coupled to the operating link 64, while the pivot pin 70 is pivotally coupled to the follower link 66 such that the first operating member 60 moves along the first arc A₁. The actuating portion 60a can have a button 73 coupled thereto as seen in FIG. 2, and is arranged to be actuated or pushed by a thumb or finger of the rider.

The second operating member 62 is preferably a thin flat member constructed of lightweight rigid material such as deformed sheet metal. The second operating member 62 is slidably and pivotally mounted on the secondary fixing bolt 42. More specifically, the second operating member 62 is pivotally and slidably mounted on a spacer 63 that is mounted on the second fixings bolt 42 between the second operating member 62 and the follower link 66. The second operating member 62 basically includes an elongated opening 62a, an actuating flange 62b and a release flange 62c. The elongated opening 62a has an upper end of the spacer 63 received therein. The release flange 62c is received in the guide opening 36a₄ of the intermediate plate 36 to control movement of the second operating member 62 together with the spacer 63, which is listed.

The second operating member 62 is actuated by a tap lever 71 that is pivotally coupled to the mounting flanges 36c₁ and 36c₂ of the intermediate plate 36. More specifically, the tap lever 71 is operatively coupled to the actuating flange 62b such that when the rider pushes the tap lever 71 with a thumb or finger, the second operating member 62 moves in a substantially linear direction along a second arc A₂ as best seen in FIG. 38. More specifically, the

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release flange 62c is received in the guide opening 36a₄ of the intermediate plate 36, while the upper end of the spacer 63 is received in the elongated opening 62a such that the actuating flange 62b moves along the arc A₂.

The arc A₂ is substantially parallel to the arc A₁ and both arcs preferably have large radii of curvature such that substantially linear parallel movement between the first and second operating members 60 and 62 is achieved. A return biasing member or spring 72 is operatively coupled between the second operating member 62 and the intermediate plate 36. Thus, when the rider releases the tap lever 71, the second operating member 62 and the tap lever 71 return to a normal rest position. The guide opening 36a₄ of the intermediate plate 36 and the release flange 62c are configured to selectively engage/actuate the retaining mechanism 30 as discussed below.

The operating link 64 is preferably a thin flat step-shaped member constructed of a lightweight rigid material such as deformed sheet metal. More specifically, the operating link 64 basically includes an operating portion 64a and a coupling portion 64b. As mentioned above, the operating link 64 is pivotally coupled to the main fixing bolt 40 and the pivot pin 69, which is pivotally coupled to the first operating member 60. The operating portion 64a basically includes a main hole 64a₁, a pawl mounting hole 64a₂, a projection 64a₃, a cutout 64a₄ and a spring mounting hole 64a₅.

A spacer 65 is mounted on the main fixing bolt 40. The spacer 65 has a step-shaped exterior surface with a bottom end mounted in the main hole 64a₁ of the operating link 64. The upper end of the spacer 65 contacts the intermediate plate 36 and surrounds the main hole 36a₁ of the intermediate plate 36. Thus, the operating portion 64a of the operating link 64 is spaced from the intermediate plate 36 and rotatably mounted on the spacer 65.

The pawl mechanism 68 is coupled to the pawl mounting hole 64a₂ and basically includes a pawl pivot pin 74, a pawl spring 76 and a pawl member 78. The pawl pivot pin 74 has a lower end mounted in the pawl mounting hole 64a₂ of the operating link 64. The pawl member 78 is mounted on the upper end of the pawl pivot pin 74 with the pawl spring 76 normally biasing the pawl member 78 relative to the operating link 64. Specifically, the pawl spring 76 has an upper end 76a coupled to the pawl member 78, a lower end 76b received in a spring mounting hole 64a₅ of the operating link 64 and a coiled portion 76c surrounding the pawl pivot pin 74. The projection 64a₃ and the cutout 64a₄ of the operating link 64 act as stop members or movement controlling members of the operating link 64. More specifically, the projection 64a₃ and the cutout 64a₄ are configured such that portions of the operating link 64 do not interfere with the movement of other members of the front shift operating device 20 and/or control movement of the operating link 64.

The pawl member 78 basically includes a mounting portion 78a and a pawl portion 78b. The mounting portion 78a is thinner than the pawl portion 78b and is received on the pawl pivot pin 74. The pawl portion 78b is configured to selectively contact the operating surface 54b of the ratchet member 54. Specifically, when the operating link 64 is moved by the first operating member 60, the pawl portion 78b engages one of the grooves 54b₁ of the operating surface 54b to rotate the ratchet member 54 and the winding member 52 about the main fixing bolt 40. However, when the first operating member 60 and the operating link 64 are in a normal rest position, the pawl portion 78b contacts the outer release surface 36a₅ of the intermediate plate 36.

In other words, the outer release surface 36a₅ is shaped to hold the pawl portion 78b out of engagement with the

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grooves 54b₁ of the operating surface 54b when the operating link 64 and the first operating member 60 are in their normal rest positions. Thus, if the second operating member 62 is actuated by the tap lever 71, e.g., if the retaining mechanism 30 is released as discussed below, the ratchet member 54 and the winding member 52 can rotate due to the urging force of the main biasing member 56.

The coupling portion 64b of the operating link 64 is offset from the operating portion 64a and basically includes a coupling hole 64b₁. The coupling hole 64b₁ is configured to receive the tower end of the pivot pin 69 that is pivotally coupled to the first operating member 60.

Referring again to FIGS. 4 and 19–21, the follower link 66 basically includes an operating portion 66a, a coupling portion 66b and a protrusion 66c. The follower link 66 is preferably a step-shaped thin flat member constructed of lightweight rigid material such as deformed sheet metal. The operating portion 66a is pivotally coupled to the pivot pin 70 that is pivotally coupled to the first operating member 60. The coupling portion 66b is pivotally coupled to the secondary fixing bolt 42 via the spacer 63. The operating portion 66a is offset from the coupling portion 66b. The protrusion 66c extends upwardly from the coupling portion 66b and is configured to receive one end of a return biasing member (spring) 79.

More specifically, the operating portion 66a includes a hole 66a₁ configured to rotatably receive a lower end of the pivot pin 70 while the coupling portion 66b includes a hole 66b₁ configured to receive the lower end of the spacer 63. The return biasing member 79 has one end coupled to the protrusion 66c and the other end coupled to the intermediate plate 36 to normally urge the follower link 66 to the rest position. Thus, the first operating member 60 and the first operating link 64 are also normally biased to the rest position by the return biasing member (spring) 79 since these members form three links of a four-bar linkage assembly.

Referring now to FIGS. 4, 6 and 28, the retaining mechanism 30 will now be discussed in more detail. The retaining mechanism 30 basically includes a retaining pivot pin 80, a retaining biasing member 82 and a locking member 84. The retaining pivot pin 80 is coupled to the base plate 34 and the intermediate plate 36. The locking member 84 is rotatably mounted on the retaining pivot pin 80. The retaining biasing member 82 normally biases the locking member 84 toward a predetermined position. The locking member 84 is arranged and configured to engage the positioning surface 54c of the ratchet member 54. Moreover, the locking member 84 is also arranged and configured to selectively engage the release flange 62c of the second operating member 62.

More specifically, the retaining pivot pin 80 has an upper end received in the mounting bore 34b of the base plate 34, and a lower end received in the positioning hole 36a₃ of the intermediate plate 36. Thus, the retaining pivot pin 80 and the base plate 34 are non-movable relative to the main mounting portion 32. The locking member 84 is arranged on the lower end of the retaining pivot pin 80 adjacent the guide opening 36a₄ of the intermediate plate 36. The biasing member 82 has an upper end engaged with the projecting section 34d of the base plate 34 and a lower end engaged with the locking member 84 to normally bias the locking member 84 in a predetermined direction.

The locking member 84 basically includes a first stop portion 84a, a second stop portion 84b, a central hole 84c, a spring receiving recess 84d and an actuating projection 84e. The lower end of the retaining pivot pin 80 is received in the hole 84c such that the locking member 84 is located

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between the base plate **34** and the intermediate plate **36**. The first and second stop portions **84a** and **84b** are arranged on opposite lateral sides of the teeth **54c₁** of the ratchet member **54**. The spring receiving recess **84d** is configured to receive the lower end of the biasing member **82**. The actuating projection **84e** is arranged adjacent the release flange **62c** of the second operating member **62** that is received in the guide opening **36a₄**.

When the second operating member **62** is actuated by the tap lever **71**, the release flange **62c** engages the actuating projection **84e** to rotate the locking member **84** against the biasing force of the spring **82**. This moves the first stop portion **84a** out of engagement with the positioning surface **54c**. Thus, the ratchet member **54** and the winding member **52** are rotated one shift position due to the biasing force of the main biasing member or spring **56**. When the tap lever **71** is released, the locking member **84** rotates via the biasing force of the spring **82** to engage the next tooth of the positioning surface **54c** to hold the ratchet member **54** and the winding member **52** in the desired shift position.

Operation of the front shift operating device **20** will now be discussed in more detail with reference to FIGS. **4**, **6** and **29-40**. When the rider wishes to shift the front derailleur **15** and the chain **C** from a smaller front sprocket **FS** to a larger front sprocket **FS**, the rider pushes the first operating member **60** with a thumb or finger. The first operating member **60** moves along the arc **A₁** to the shift position. This movement of the first operating member **60** causes both the operating link **64** and the follower link **66** to rotate about the main and secondary fixing bolts **40** and **42**, respectively. When the operating link **64** rotates about the main fixing bolt **40**, the pawl mechanism **68** moves with the operating link **64**.

When the pawl mechanism **68** moves, the pawl member **78** moves out of engagement with the release surface **36a₅** of the intermediate plate **36**. The pawl member **78** then engages the ratchet member **54** to rotate the ratchet member **54** and the winding member **52** one shift position. When the ratchet member is rotated from one shift position to the next adjacent shift position by the pawl mechanism **68**, the locking member **84** is moved out of engagement with its respective tooth **54c₁** and into engagement with the next respective tooth **54c₁**. Specifically, as the ratchet member **54** moves between two shift positions, the locking member **84** disengages and then reengages the teeth **54c₁** to retain the ratchet member **54** in the next shift position.

When the first operating member **60** is released by the rider, the first operating member **60** moves back along the first arc **A₁** to its normal rest position. This movement of the first operating member **60** causes the operating link **64** and the follower link **66** to rotate about the main and secondary fixing bolts **40** and **42**, respectively. The operating link **64** and the follower link **66** rotate until they return to their normal rest positions. The pawl mechanism **78** moves with the operating link **64**. Thus, the pawl member **78** moves back into engagement with the release surface **36a₅** so that the pawl member **78** is out of engagement with the operating surface **54b** of the ratchet member **54**. However, the ratchet member **54** is retained in its shift position due to the locking member **84**. If the rider desires to shift from a smaller front sprocket **FS** to a larger front sprocket **FS** again, the first operating member **60** is pushed again along the first arc **A₁** and the process described above is repeated.

If the rider desires to shift the front derailleur **15** and the chain **C** from a larger front sprocket **FS** to a smaller front sprocket **FS**, the rider pushes the tap lever **71** with a thumb or finger. When the tap lever **71** is actuated by the rider, the

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second operating member **62** moves along the second arc **A₂** from the normal rest position to the shift position. Specifically, movement of the second operating member **62** is controlled by the guide opening **36a₄** of the intermediate plate and the spacer **63**, which is coupled to the secondary fixing bolt **42**. When the second operating member **62** is moved to the shift position, the release flange **62c** engages the locking member **84**. The locking member **84** is rotated against the biasing force of the spring **82** out of engagement with the teeth **54c₁** of the ratchet member **54**. The ratchet member **54** can then rotate under the biasing force of the main biasing member or spring **56**.

The locking member **84** will then engage the next adjacent tooth **54c₁** upon releasing the second operating member **62**. Specifically, when the rider pushes the tap lever **71**, the locking member **84** is disengaged from the teeth **54c₁**. However, when the rider then releases the tap lever **71**, the second operating member **62** will return to its normal rest position under the biasing force of the return spring **72**. Thus, the release flange **62c** will release the locking member **84**. In other words, the second operating member **62** is designed to be tapped and released relatively quickly by the rider. The locking member **84** then non-rotatably engages the next adjacent tooth **54c₁** of the ratchet member **54** and the shift from the larger front sprocket **FS** to the smaller front sprocket **FS** is complete. This process can then be repeated by the rider if desired.

Of course, if the chain **C** is already on the smallest front sprocket **FS**, the stops between the locking member **84** and the ratchet member **54** will prevent disengagement with the teeth **54c₁**. Therefore, if the rider pushes the tap lever **71** when the front derailleur **15** is positioned over the smallest front sprocket **FS**, a shift will not occur. Likewise, if the front derailleur **15** is in its top position such that the chain **C** is arranged on the largest front sprocket **FS**, the first operating member **60** cannot be shifted due to the stops between the locking member **84** and the ratchet member **54**. Thus, a shift will not occur in this situation.

Rear Shift Operating Device

Referring now to FIGS. **3** and **5**, the rear shift operating device **22** will now be discussed in more detail. The rear shift operating device **22** is substantially identical to the front shift operating device **20** except that certain parts of the rear shift operating device have been modified so that the rear shift operating device **22** has more shift positions than the front shift operating device **20**. Specifically, the rear shift operating device **22** is designed to include more shift positions than the front shift operating device **20** because the rear derailleur **17** has more shift positions than the front derailleur **15**. However, the rear shift operating device **22** utilizes operating members that have the same or almost the same stroke as the front shift operating device **20**, as discussed below. In other words, the rear shift operating device **22** is configured to be used with the rear derailleur **17**, which is designed to shift between a number of rear sprockets **RS** larger than the number of front sprockets **FS**.

The rear shift operating device **22** basically includes a mounting assembly **124**, a winding mechanism **126**, an operating mechanism **128** and a retaining mechanism **130** coupled together to shift the rear derailleur **17** and the chain **C** of the drive train or transmission **14** between the rear sprockets **RS**. In the illustrated embodiment, the transmission **14** preferably has more than three rear sprockets **RS** coupled thereto. More specifically, in the illustrated embodiment, the transmission preferably has nine rear

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sprockets RS. Thus, the rear shift operating device 22 also preferably has more than three shift positions. More specifically, the rear shift operating device 22 preferably has nine shift positions. In any event, the rear shift operating device 22 preferably has a number of shift positions that corresponds to the number of shift positions of the rear derailleur 17. Of course, it will be apparent to those skilled in the art from this disclosure that the rear derailleur 17 and the rear shift operating device 22 could be designed with a different number of shift positions if needed and/or desired.

The winding mechanism 126, the operating mechanism 128 and the retaining mechanism 130 are coupled to the mounting assembly 124. The winding mechanism 126 is controlled by the operating mechanism 128 and the retaining mechanism 130 to selectively maintain the rear derailleur 17 in one of a plurality of shift positions via the control cable 23. More specifically, the operating mechanism 128 and the retaining mechanism 130 control rotation of the winding mechanism 126 to selectively retain the winding mechanism 126 in one of the plurality of shift positions. Thus, the inner wire of the control cable 23 and the rear derailleur 17 are also selectively retained in one of a plurality of shift positions. The inner wire of the rear control cable 23 is coupled to the winding mechanism 126 such that rotation of the winding mechanism 126 takes-up or lets-out the inner wire of the control cable 23 to actuate/move/shift the rear derailleur 17 between the rear sprockets RS.

The mounting assembly 124 basically includes a plurality of fixed members coupled together to form a shift operating device housing that supports the winding mechanism 126, the operating mechanism 128 and the retaining mechanism 130. More specifically, the mounting assembly 124 basically includes a main mounting portion 132, a base plate 134, an intermediate plate 136 and a lever retaining plate 138. The base plate 134, the intermediate plate 136 and the lever retaining plate 138 are basically fixedly coupled to the main mounting portion 132 by a main fixing bolt 140 (first fixed pivot axle) and a secondary fixing bolt 142 (second fixed pivot axle) that are preferably parallel to each other to form the shift operating device housing. Various other parts of the rear shift operating device 22 (i.e., parts of the winding mechanism 126, operating mechanism 128 and retaining mechanism 130) are either movably or non-movably coupled to the mounting assembly 124, as discussed below in more detail.

The main mounting portion 132 basically has a main/shift mounting plate 132a, a handlebar mounting bracket 132b and a brake lever mounting bracket 132c as best seen in FIGS. 3 and 5. A brake lever 133 is pivotally coupled to the brake lever mounting bracket 132c of the main mounting portion 132 in the illustrated embodiment. The brake lever 133 is operatively coupled to the rear brake 19b by a bowden brake cable in a conventional manner. Preferably, the main/shift mounting plate 132a, the handlebar mounting bracket 132b and the brake lever mounting bracket 132c are integrally formed together as a one-piece unitary member that is coupled to the handlebar 13 via the mounting bracket 132b. Thus, an integral rear shift/brake operating device 22 is preferably provided. The main mounting portion 132 is preferably constructed of a lightweight rigid material such as cast aluminum. Of course, any suitable material could be utilized if needed and/or desired.

The base plate 134, the intermediate plate 136 and the retaining plate 138 are preferably fixedly coupled to the main/shift mounting plate 132a via the fixing bolts 140 and 142 in a spaced arrangement to accommodate parts of the winding mechanism 126, operating mechanism 128 and

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retaining mechanism 130 therebetween. More specifically, the main/shift mounting plate 132a includes a main through hole 132a₁ and blind threaded bore 132a₂ configured to mount the fixing bolts 140 and 142 therein, respectively. A cover 144 and indicating mechanism 146 are also preferably fixedly coupled to the main/shift mounting plate 132a. However, the cover 144 and the indicating mechanism 146 are relatively conventional. Thus, the cover 144 and the indicating mechanism 146 will not be discussed and/or illustrated in detail herein. Moreover, the cover 144 and the indicating mechanism 146 are omitted from some Figures for the purpose of clarity. In any event, the indicating mechanism 146 is preferably operatively coupled with the shift operating device 22 to indicate the various shift positions in a conventional manner.

As seen in FIGS. 5 and 41, the base plate 134 is a relatively thin, flat member. The base plate 134 is preferably constructed of a lightweight rigid material such as sheet metal. The base plate 134 is located between the main mounting portion 132 and the intermediate plate 136 and basically has a main through bore 134a, a positioning mounting bore 134b, a spring mounting slot 134c, a projecting section 134d and a T-shaped opening 134e. The main through bore 134a is configured to receive the main fixing bolt 140 therethrough, while the mounting bore 134b is configured to receive part of the retaining mechanism 130. Specifically, the retaining mechanism 130 is coupled to the positioning mounting bore 134b and the intermediate plate 136 to prevent rotation of the base plate 134 relative to the main mounting portion 132 as discussed below. Also, a portion of the intermediate plate 136 extends partially through the T-shaped opening 134e to prevent rotation of the base plate 134, and acts as a spacer as also discussed below. The spring mounting slot 134c is configured to engage a portion of the winding mechanism 126 as discussed below. The projecting section 134d is configured to engage a portion of the retaining mechanism 130 as also discussed below.

As seen in FIGS. 5 and 42, the intermediate plate 136 is preferably formed of several thin flat portions integrally formed together as a one-piece unitary member. Moreover, the intermediate plate 136 is preferably constructed of a lightweight rigid material such as deformed sheet metal. The intermediate plate 136 is located between the base plate 134 and the lever retaining plate 138, and basically has a base portion 136a, a connecting portion 136b, and a lever mounting portion 136c. The base portion 136a is fixedly and non-rotatably coupled to the main/shift plate 132a. The connecting portion 136b extends outwardly from the base portion 136a. The lever mounting portion 136c is coupled to the free end of the connecting portion 136b.

The base portion 136a basically includes a main through hole 136a₁, a secondary hole 136a₂, a positioning hole 136a₃, a guide opening 136a₄ and a release surface 136a₅. A flange 136a₆ with a mounting hole 136a₇ preferably extends from the base portion 136a to form a right angle. An auxiliary hole 136a₈ is arranged between the main through hole 136a₁ and the release surface 136a₅. The main hole 136a₁ is configured to receive the main fixing bolt 140 therethrough while the secondary hole 136a₂ is configured to receive the secondary fixing bolt 142 therethrough. Thus, the base portion 136a is non-rotatably coupled to the main mounting portion 132. The retaining mechanism 130 is coupled to the positioning hole 136a₃ and the hole 134b such that the base plate 134 is non-rotatably coupled to the main mounting portion 132 via the base portion 136a, as discussed below.

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The guide opening **136a₄** is configured to receive a portion of the operating mechanism **128** as discussed below in more detail. The release surface **136a₅** is configured to selectively engage a portion of the operating mechanism **128** as also explained below. The flange **136a₆** extends partially through the T-shaped opening **134e** of the base plate **134** to non-rotatably couple the base plate **134** to the intermediate plate **136**. The flange **136a₆** also acts as a spacer between the base plate **134** and the intermediate plate **136** due to the stepped configuration of the flange **136a₆**. The mounting hole **136a₇** receives a fastener **145** to couple the control cable **23** thereto. The auxiliary hole **136a₈** is configured to receive a pivot pin **148** (auxiliary, fixed pivot axle) therein. A portion of the operating mechanism **128** is rotatably mounted or coupled on the pivot pin **148** as explained below. The pivot pin **148** is preferably arranged to be parallel to the main fixing bolt **40** (first pivot axle).

The lever mounting portion **136c** includes first and second mounting flanges **136c₁** and **136c₂**, respectively. The mounting flanges **136c₁** and **136c₂** include holes **136c₃** and **136c₄**, respectively, that are configured to receive a portion of the operating mechanism **128**, as also discussed below in more detail. The connecting portion **136b** extends between the lever mounting portion **136c** and the base portion **136a** such that the lever mounting portion **136c** is spaced outwardly from the base portion **136a**.

As seen in FIGS. **5** and **43**, the lever retaining plate **138** is preferably a thin flat member. The lever retaining plate **138** is preferably constructed of a lightweight rigid material such as sheet metal. The lever retaining plate **138** is located on an opposite side of the intermediate plate **136** from the base plate **134**. Thus, the lever retaining plate **138** is located furthest from the main mounting portion **132**. The lever retaining plate **138** is fixedly and non-rotatably coupled to the main/shift mounting plate **132a** via the fixing bolts **140** and **142**. The lever retaining plate **138** basically includes a main support portion **138a** and an extending portion **138b**.

The main support portion **138a** has a main hole **138a₁**, a pair of opposed auxiliary holes **138a₂**, and a pair of opposed retaining holes **138a₃**. The main hole **138a₁** is configured to receive the main fixing bolt **140** therethrough. In this embodiment, one of the auxiliary holes **138a₂** is used to receive a lower end of the pivot pin **148**. In other words, the lever retaining plate **138** is identical to the lever retaining plate **38** but in the rear shift operating device **22**, one of the auxiliary holes **138a₂** has the pivot pin **148** partially received therein. The pair of retaining holes **138a₃** are configured to receive projections of a main nut plate **141** to prevent loosening of the main fixing bolt **140** after the parts of the mounting assembly **124** are coupled together.

The extending portion **138b** includes a secondary hole **138b₁** and a pair of retaining holes **138b₂**. The secondary hole **138b₁** is configured to receive the secondary fixing bolt **142** therethrough. The retaining holes **138b₂** are configured to receive projections of a secondary bolt plate **143** to prevent loosening of the secondary fixing bolt **142** after the parts of the mounting assembly **124** are coupled together. Thus, the lever retaining plate **138**, the intermediate plate **136**, the base plate **134** and the main/shift plate **132a** are fixedly and non-rotatably coupled together to form the housing or mounting assembly **124**.

Referring to FIGS. **5** and **44-47**, the winding mechanism **126** will now be discussed in more detail. The winding mechanism **126** is mounted on the main fixing bolt **140** between the base plate **134** and the intermediate plate **136** of the mounting assembly **124**. The winding mechanism **126**

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basically includes a cable winding member **152**, a ratchet member **154**, a main biasing member **156** and a unit spacer **158**. In the illustrated embodiment, the winding member **152** and the ratchet member **154** are preferably separate members that are non-rotatably coupled together. The main biasing member **156** is preferably a torsion spring that normally biases the winding member **152** and the ratchet member **154** in a predetermined rotational direction, e.g., the clockwise direction CW as seen in FIG. **5**. The winding mechanism **126** is operatively coupled to the operating mechanism **128** and the retaining mechanism **130** to selectively retain the winding member **152** in one of a plurality of shift positions.

The winding member **152** is preferably a step-shaped member integrally formed as a one-piece unitary member. Moreover, the winding member **152** is preferably constructed of a lightweight rigid material such as hard plastic. The winding member basically includes a winding portion **152a** and a projecting portion **152b**. A central through opening **152a₁** extends through both the winding portion **152a** and the projecting portion **152b** and is configured to receive the main fixing bolt **140** therethrough. Thus, the winding member **152** is coupled to the mounting assembly **124** for rotation about the main fixing bolt **140**. The ratchet member **154** is preferably non-rotatably coupled to the winding member **152** and is located between the winding member **152** and the intermediate plate **136**. Thus, the ratchet member **154** rotates with the winding member **152**. As explained below, the rotational movement of the winding member **152** is controlled by the ratchet member **152**.

The winding portion **152a** includes a cable mounting section **152a₂**, a winding surface **152a₃** and a spring receiving recess **152a₄**. The cable mounting section **152a₂** includes a through opening configured to receive the inner wire of the control cable **23** and a recess configured to receive a nipple or abutment of the inner wire of the control cable **23** in a conventional manner. The winding surface **152a₃** is a curved groove-shaped surface which extends circumferentially away from the cable mounting section **152a₂**. The spring receiving recess **152a₄** is a substantially annular recess with a larger diameter than the central through opening **152a₁**. Moreover, the spring receiving recess **152a₄** includes a cutout or notch **152a₅** configured to receive one end of the spring **156** as discussed below. The spring receiving recess **152a₄** preferably has a radial width sufficient to receive a portion of the main spring **156** as also discussed below in more detail. The winding portion **152a** also preferably includes a stepped groove **152a₆** configured to receive a portion (not shown) of the indicating mechanism **146**.

The projecting portion **152b** basically includes a plurality of large splines **152b₁** and a plurality of small splines **152b₂** that are configured to non-rotatably engage the ratchet member **154**. The large splines **152b₁** are arranged on an opposite side of the central through opening **152a₁** from the small splines **152b₂**. Moreover, the splines **152b₁** and **152b₂** are oriented such that the ratchet member **154** is arranged in a certain configuration in order to properly engage both the operating mechanism **128** and the retaining mechanism **130**, as discussed below.

The ratchet member **154** is preferably a thin flat member. Preferably, the ratchet member **154** is constructed of a lightweight rigid material such as sheet metal. The ratchet member **154** basically includes a splined central opening **154a**, an outer operating surface **154b** and an outer positioning surface **154c**. The opening **154a** is configured to receive the projecting portion **152b** of the winding member **152** to non-rotatably couple the ratchet member **154** to the

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winding member 152 in a predetermined orientation. In other words, the opening 154a mates with the large splines 152b₁ and the small splines 152b₂ of the projecting portion 152b so that the winding member 152 rotates with the ratchet member 154 as if they were a single piece.

The operating surface 154b is configured to engage a portion of the operating mechanism 128, which is discussed below. More specifically, the operating surface 154b includes a plurality (nine) of grooves 154b₁ that form teeth designed to selectively engage a portion of the operating mechanism 128. The positioning surface 154c is configured to engage a portion of the retaining mechanism 130 (which is also discussed below) to selectively hold and retain the ratchet member 154 and the winding member 152 in one of the plurality of shift positions. More specifically, the positioning surface 154c includes a plurality (nine) of projections or teeth 154c₁, which engage a portion of the retaining mechanism 130. Thus, the ratchet member 154 is a modified version of the ratchet member 54 in order to provide more shift positions. The ratchet member 154 also preferably includes a pair of stops 154d and 154e. Specifically, the stop 154d is a rest stop while the stop 154e is a fully shifted stop. The stops 154d and 154e selectively engage portions of the retaining mechanism 130 to control movements of the winding member 152 and the ratchet member 154. The ratchet member 154 is configured to have a smaller operating angle for each shift between an adjacent pair of shift positions than the ratchet member 54. Thus, the operating mechanism 128 is a modified version of the operating mechanism 28 in order to cooperate with the ratchet members 154 as explained below.

The biasing member 156 is preferably a torsion spring that is arranged to apply an urging force on the winding member 152 and the ratchet member 154 to rotate about the main fixing bolt 140. The biasing member (spring) 156 basically includes a first end 156a, a second end 156b and a coiled portion 156c extending between the first and second ends 156a and 156b. The first end 156a is arranged in the spring mounting slot 134c of the base plate 134, while the second end 156b is arranged in the notch or cutout 152a₅ extending from the spring receiving recess 152a₄ of the winding member 152. The coiled portion 156c is partially received in the spring receiving recess 152a₄. The unit spacer 158 is disposed in the central through opening 152a₁ of the winding member 152.

The unit spacer 158 is preferably a tubular member constructed of lightweight rigid material with one end contacting the lower surface of the base plate 134 and the other end contacting the upper surface of the intermediate plate 136 to space the intermediate plate 136 from the base plate 134. The spring 156 is configured to normally bias the winding member 152 in a clockwise direction as viewed from above in FIG. 5. Thus, the operating mechanism 128 moves the winding member 152 via the ratchet member 154 in a counter-clockwise direction against the urging force of the main biasing member 156. More specifically, the operating mechanism 128 preferably moves the winding member 152 to one of nine predetermined shift positions, as discussed below. The retaining mechanism 130 selectively retains the winding member 152 and the ratchet member 154 in one of the nine shift positions, as also discussed below.

Referring now to FIGS. 5 and 49–61, the operating mechanism 128 will now be discussed in more detail. The operating mechanism 128 is a modified version of the operating mechanism 28 in order to accommodate the larger number of shift positions of the ratchet member 54 as explained below. The operating mechanism 128 basically

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includes a first operating member 160, a second operating member 162, an operating link 164 and a follower link 166. The first operating member 160 together with the operating link 164, the follower link 166 and the mounting assembly 124 form a four-bar linkage that controls movement of the first operating member 160. More specifically, the first operating member 160 is pivotally coupled to both the operating link 164 and the follower link 166 to move along a first arc 1A₁ as best seen in FIGS. 63 and 64. The operating link 164 is pivotally coupled on the pivot pin 148 (auxiliary fixed pivot axle) and operatively coupled on the main fixing bolt 140, while the follower link 166 is pivotally coupled to the secondary fixing bolt 142 (second fixed pivot axle). The first arc 1A₁ has a relatively flat curvature such that the first operating member 160 moves substantially in a straight line.

The operating link 164 preferably has a pawl mechanism 168 operatively coupled thereto. The pawl mechanism 168 is configured to engage the operating surface 154b of the ratchet member 154 when the first operating member 160 is pushed or moved by the rider from the normal rest position to the shift position. Thus, when the first operating member 160 is moved in a substantially linear direction along the arc 1A₁, the operating link 164 is also moved such that the pawl mechanism 168 rotates the ratchet member 154 one shift position against the urging force of the biasing member 156. Thus, the winding member 152 is also rotated to pull the inner wire of the shift cable 23 and shift the rear derailleur 17 to one of its shift positions.

When the first operating member 160 moves back to the normal rest position, the pawl mechanism 168 engages the release surface 136a₅ of the intermediate plate 136 to disengage the pawl mechanism from the ratchet member 154. However, the retaining mechanism 130 engages the ratchet member 154 to hold the winding member 152 and the ratchet member 154 in the new shift position against the urging force of the spring 156. In the illustrated embodiment, movement of the first operating member 160 causes only one shift of the winding member 152 from one shift position to the next adjacent shift position in one rotational direction.

The second operating member 162 is operatively coupled to the retaining mechanism 130 to selectively release the ratchet member 154 and the winding member 152. In other words, the rider pushes or moves the second operating member 162 to disengage the retaining mechanism 130 from the ratchet member 154, which allows the winding member 152 and the ratchet member 154 to rotate under the urging force of the spring 156. The retaining mechanism 130 is configured and arranged so that the winding member 152 and the ratchet member 154 rotate only one shift position for each push of the second operating member 162. In other words, the ratchet member 154 engages the retaining mechanism 130 after rotating one shift position. Thus, in the illustrated embodiment, movement of the second operating member 162 causes only one shift of the winding member 152 from one shift position to the next adjacent shift position in another rotational direction opposite to the rotational direction caused by moving the first operating member 160.

As best seen in FIGS. 5 and 49, the first operating member 160 is a thin flat member. The first operating member 160 is preferably constructed of lightweight rigid material such as deformed sheet metal. The first operating member 160 basically has an actuating portion 160a and a link portion 160b. The link portion 160b is preferably perpendicular to the actuating portion 160a. The link portion 160b includes a pair of holes 160b₁ and 160b₂ that are configured to receive the upper ends of a pair of pivot pins 169 and 170, respec-

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tively. The pivot pin **169** is pivotally coupled to the operating link **164**, while the pivot pin **170** is pivotally coupled to the follower link **166** such that the first operating member **160** moves along the first arc $1A_1$. The actuating portion **160a** can have a button **173** coupled thereto as seen in FIG. 3, and is arranged to be actuated or pushed by a thumb or finger of the rider.

The second operating member **162** is preferably a thin flat member constructed of lightweight rigid material such as deformed sheet metal. The second operating member **162** is slidably and pivotally mounted on the secondary fixing bolt **142**. More specifically, the second operating member **162** is pivotally and slidably mounted on a spacer **163** that is mounted on the second fixing bolt **142** between the second operating member **162** and the follower link **166**. The second operating member **162** basically includes an elongated opening **162a**, an actuating flange **162b** and a release flange **162c**. The elongated opening **162a** has an upper end of the spacer **163** received therein. The release flange **162c** is received in the guide opening **136a₄** of the intermediate plate **136** to control movement of the second operating member **162** together with the spacer **163**, which is fixed.

The second operating member **162** is actuated by a tap lever **171** that is pivotally coupled to the mounting flanges **136c₁** and **136c₂** of the intermediate plate **136**. More specifically, the tap lever **171** is operatively coupled to the actuating flange **162b** such that when the rider pushes the tap lever **171** with a thumb or finger, the second operating member **162** moves in a substantially linear direction along a second arc (not shown) in manner substantially identical to the second operating member **62**. More specifically, the release flange **162c** is received in the guide opening **136a₄** of the intermediate plate **136**, while the upper end of the spacer **163** is received in the elongated opening **162a** such that the actuating flange **162b** moves along the second arc.

The second arc is substantially parallel to the arc $1A_1$ and both arcs preferably have large radii of curvature such that substantially linear parallel movement between the first and second operating members **160** and **162** is achieved. In other words, the arc $1A_1$ and the second arc are basically mirror images of the arcs A_1 and A_2 of the shift operating device **20**. A return biasing member or spring **172** is operatively coupled between the second operating member **162** and the intermediate plate **136**. Thus, when the rider releases the tap lever **171**, the second operating member **162** and the tap lever **171** return to a normal rest position. The guide opening **136a₄** of the intermediate plate **136** and the release flange **162c** are configured to selectively engage/actuate the retaining mechanism **130** as discussed below.

The operating link **164** is preferably a thin flat step-shaped member constructed of a lightweight rigid material such as deformed sheet metal. More specifically, the operating link **164** basically includes an operating portion **164a** and a coupling portion **164b**. The coupling portion **164b** can be considered a first portion of the operating link **164**, while the operating portion **164a** can be considered a second portion of the operating link **164**. As mentioned above, the second or operating portion **164a** of the operating link **164** is pivotally coupled to the pivot pin **148** (auxiliary pivot axle) and operatively coupled to the main fixing bolt **140** (first pivot axle), and the first or coupling portion **164b** of the operating link **164** is pivotally coupled to the pivot pin **169**, which is pivotally coupled to the first operating member **160**. Thus, the first or coupling portion **164b** of the operating link **164** is movably coupled to the first operating member **160**. The operating portion **164a** basically includes a main hole **164a₁**, a pawl mounting hole **164a₂**, a projection **164a₃**, and

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an auxiliary hole **164a₄**. The operating link **164** is a modified version of the operating link **64** and moves differently than the operating link **64**, as explained below.

A spacer **165** is mounted on the main fixing bolt **140** and is fixed. The spacer **165** has a step-shaped exterior surface with a bottom end mounted in the main hole **164a₁** of the operating link **164**. The main hole **164a₁** is shaped as an elongated circumferential guide opening larger than the lower end of the spacer **165**. Thus, the operating link is movable along a transverse arc relative to the main fixing bolt **140**. Specifically, the lower end of the spacer **165** slides relative to the operating portion **164a** due to the configuration of elongated circumferential guide opening **164a₁**. The upper end of the spacer **165** contacts the intermediate plate **136** and surrounds the main hole **136a₁** of the intermediate plate **136**. Thus, the operating portion **164a** of the operating link **164** is spaced from the intermediate plate **136**.

The pivot pin **148** is fixed between the intermediate plate **136** and the lever retaining plate **138**. Moreover, the pivot pin **148** extends through the auxiliary hole **164a₄** such that the operating link **164** pivots about the pivot pin **148**, not the main fixing bolt **140**. Thus, the operating link **164** moves differently than the operating link **64**. More specifically, the auxiliary hole **164a₄** is closer to the pawl mounting hole **164a₂** than the main hole **164a₁** such that the pawl mechanism **168** moves a smaller operating angle than the pawl mechanism **68**. In other words, the operating link **164** and the pawl mechanism **168** are configured to cooperate with the ratchet member **154**, which has smaller operating angles between the various shift positions such that the first operating member **160** has a stroke substantially identical to the stroke of the first operating member **60** of the front shift operating device **20**.

The pawl mechanism **168** is coupled to the pawl mounting hole **164a₂** and basically includes a pawl pivot pin **174**, a pawl spring **176** and a pawl member **178**. The pawl pivot pin **174** has a lower end mounted in the pawl mounting hole **164a₂** of the operating link **164**. The pawl member **178** is mounted on the upper end of the pawl pivot pin **174** with the pawl spring **176** normally biasing the pawl member **178** relative to the operating link **164**. Specifically, the pawl spring **176** has an upper end **176a** coupled to the pawl member **178**, a lower end **176b** received in a spring mounting hole **164a₅** of the operating link **164** and a coiled portion **176c** surrounding the pawl pivot pin **174**. The projection **164a₃** of the operating link **164** acts as stop member or movement controlling member of the operating link **164**. More specifically, the projection **164a₃** is configured such that portions of the operating link **164** do not interfere with the movement of other members of the rear shift operating device **22** and/or control movement of the operating link **164**.

The pawl member **178** basically includes a mounting portion **178a** and a pawl portion **178b**. The mounting portion **178a** is thinner than the pawl portion **178b** and is received on the pawl pivot pin **174**. The pawl portion **178b** is configured to selectively contact the operating surface **154b** of the ratchet member **154**. Specifically, when the operating link **164** is moved by the first operating member **160**, the pawl portion **178b** engages one of the grooves **154b₁** of the operating surface **154b** to rotate the ratchet member **154** and the winding member **152** about the main fixing bolt **140**. However, when the first operating member **160** and the operating link **164** are in a normal rest position, the pawl portion **178b** contacts the outer release surface **136a₅** of the intermediate plate **136**.

In other words, the outer release surface **136a₅** is shaped to hold the pawl portion **178b** out of engagement with the

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grooves $154b_1$ of the operating surface $154b$ when the operating link 164 and the first operating member 160 are in the normal rest positions. Thus, if the second operating member 162 is actuated by the tap lever 171 , e.g., if the retaining mechanism 130 is released as discussed below, the ratchet member 154 and the winding member 152 can rotate due to the urging force of the main biasing member 156 .

The coupling portion $164b$ of the operating link 164 is offset from the operating portion $164a$ and basically includes a coupling hole $164b_1$. The coupling hole $164b_1$ is configured to receive the lower end of the pivot pin 169 that is pivotally coupled to the first operating member 160 .

Referring again to FIGS. 5 and 53–55, the follower link 166 basically includes an operating portion $166a$, a coupling portion $166b$ and a protrusion $166c$. The follower link 166 is preferably a step-shaped thin flat member constructed of lightweight rigid material such as deformed sheet metal. The operating portion $166a$ is pivotally coupled to the pivot pin 170 that is pivotally coupled to the first operating member 160 . The coupling portion $166b$ is pivotally coupled to the secondary fixing bolt 142 via the spacer 163 . The operating portion $166a$ is offset from the coupling portion $166b$. The protrusion $166c$ extends upwardly from the coupling portion $166b$ and is configured to receive one end of a return biasing member (spring) 179 .

More specifically, the operating portion $166a$ includes a hole $166a_1$ configured to rotatably receive a lower end of the pivot pin 170 while the coupling portion $166b$ includes a hole $166b_1$ configured to receive the lower end of the spacer 163 . The return biasing member 179 has one end coupled to the protrusion $166c$ and the other end coupled to the intermediate plate 136 to normally urge the follower link 166 to the rest position. Thus, the first operating member 160 and the first operating link 164 are also normally biased to the rest position by the return biasing member (spring) 179 since these members form three links of a four-bar linkage assembly.

Referring now to FIGS. 5 and 62, the retaining mechanism 130 will now be discussed in more detail. The retaining mechanism 130 basically includes a retaining pivot pin 180 , a retaining biasing member 182 and a locking member 184 . The retaining pivot pin 180 is coupled to the base plate 134 and the intermediate plate 136 . The locking member 184 is rotatably mounted on the retaining pivot pin 180 . The retaining biasing member 182 normally biases the locking member 184 toward a predetermined position. The locking member 184 is arranged and configured to engage the positioning surface $154c$ of the ratchet member 154 . Moreover, the locking member 184 is also arranged and configured to selectively engage the release flange $162c$ of the second operating member 162 .

More specifically, the retaining pivot pin 180 has an upper end received in the mounting bore $134b$ of the base plate 134 , and a lower end received in the positioning hole $136a_3$ of the intermediate plate 136 . Thus, the retaining pivot pin 180 and the base plate 134 are non-movable relative to the main mounting portion 132 . The locking member 184 is arranged on the lower end of the retaining pivot pin 180 adjacent the guide opening $136a_4$ of the intermediate plate 136 . The biasing member 182 has an upper end engaged with the projecting section $134d$ of the base plate 134 and a lower end engaged with the locking member 184 to normally bias the locking member 184 in a predetermined direction.

The locking member 184 basically includes a first stop portion $184a$, a second stop portion $184b$, a central hole

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$184c$, a spring receiving recess $184d$ and an actuating projection $184e$. The lower end of the retaining pivot pin 180 is received in the hole $184c$ such that the locking member 184 is located between the base plate 134 and the intermediate plate 136 . The first and second stop portions $184a$ and $184b$ are arranged on opposite lateral sides of the teeth $154c_1$ of the ratchet member 154 . The spring receiving recess $184d$ is configured to receive the lower end of the biasing member 182 . The actuating projection $184e$ is arranged adjacent the release flange $162c$ of the second operating member 162 that is received in the guide opening $136a_4$.

When the second operating member 162 is actuated by the tap lever 171 , the release flange $162c$ engages the actuating projection $184e$ to rotate the locking member 184 against the biasing force of the spring 182 . This moves the first stop portion $184a$ out of engagement with the positioning surface $154c$. Thus, the ratchet member 154 and the winding member 152 are rotated one shift position due to the biasing force of the main biasing member or spring 156 . When the tap lever 171 is released, the locking member 184 rotates via the biasing force of the spring 182 to engage the next tooth of the positioning surface $154c$ to hold the ratchet member 154 and the winding member 152 in the desired shift position.

Operation of the rear shift operating device 22 will now be discussed in more detail with reference to FIGS. 5 and 63–68. When the rider wishes to shift the rear derailleur 17 and the chain C from a smaller rear sprocket RS to a larger rear sprocket RS , the rider pushes the first operating member 160 with a thumb or finger. The first operating member 160 moves along the arc $1A_1$ to the shift position. This movement of the first operating member 160 causes both the operating link 164 and the follower link 166 to rotate about the pivot pin 148 and the secondary fixing bolt 42 , respectively. When the operating link 164 rotates about the pivot pin 148 , the pawl mechanism 168 moves with the operating link 164 . Also, the elongated main hole $164a_1$ slides along the spacer 165 .

When the pawl mechanism 168 moves, the pawl member 178 moves out of engagement with the release surface $136a_5$ of the intermediate plate 136 . The pawl member 178 then engages the ratchet member 154 to rotate the ratchet member 154 and the winding member 152 one shift position. When the ratchet member is rotated from one shift position to the next adjacent shift position by the pawl mechanism 168 , the locking member 184 is moved out of engagement with its respective tooth $154c_1$ and into engagement with the next respective tooth $154c_1$. Specifically, as the ratchet member 154 moves between two shift positions, the locking member 184 disengages and then reengages the teeth $154c_1$ to retain the ratchet member 154 in the next shift position.

When the first operating member 160 is released by the rider, the first operating member 160 moves back along the first arc $1A_1$ to its normal rest position. This movement of the first operating member 160 causes the operating link 164 and the follower link 166 to rotate about the pivot pin 148 and the secondary fixing bolt 42 , respectively. The operating link 164 and the follower link 166 rotate until they return to their normal rest positions. The pawl mechanism 178 moves with the operating link 164 . Thus, the pawl member 178 moves back into engagement with the release surface $136a_5$ so that the pawl member 178 is out of engagement with the operating surface $154b$ of the ratchet member 154 . However, the ratchet member 154 is retained in its shift position due to the locking member 184 . If the rider desires to shift from a smaller rear sprocket RS to a larger rear sprocket RS again, the first operating member 160 is pushed again along the first arc $1A_1$ and the process described above is repeated.

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If the rider desires to shift the rear derailleur 17 and the chain C from a larger rear sprocket RS to a smaller rear sprocket RS, the rider pushes the tap lever 171 with a thumb or finger. When the tap lever 171 is actuated by the rider, the second operating member 162 moves along the second arc 5 from the normal rest position to the shift position. Specifically, movement of the second operating member 162 is controlled by the guide opening 136a₄ of the intermediate plate and the spacer 163, which is coupled to the secondary fixing bolt 142. When the second operating member 162 is 10 moved to the shift position, the release flange 162c engages the locking member 184. The locking member 184 is rotated against the biasing force of the spring 182 out of engagement with the teeth 154c₁ of the ratchet member 154. The ratchet member 154 can then rotate under the biasing force of the 15 main biasing member or spring 156.

The locking member 184 will then engage the next adjacent tooth 154c₁ upon releasing the second operating member 162. Specifically, when the rider pushes the tap lever 171, the locking member 184 is disengaged from the 20 teeth 114c₁. However, when the rider then releases the tap lever 171, the second operating member 162 will return to its normal rest position under the biasing force of the return spring 172. Thus, the release flange 162c will release the locking member 184. In other words, the second operating member 162 is designed to be tapped and released relatively quickly by the rider. The locking member 184 then non-rotatably engages the next adjacent tooth 154c₁ of the 25 ratchet member 154 and the shift from the larger rear sprocket RS to the smaller rear sprocket RS is complete. This process can then be repeated by the rider if desired.

Of course, if the chain C is already on the smallest rear sprocket RS, the stops between the locking member 184 and the ratchet member 154 will prevent disengagement with the 30 teeth 154c₁. Therefore, if the rider pushes the tap lever 171 when the rear derailleur 17 is positioned under the smallest rear sprocket RS, a shift will not occur. Likewise, if the rear derailleur 17 is in its top position such that the chain C is arranged on the largest rear sprocket RS, the first operating member 160 cannot be shifted due to the stops between the 35 locking member 184 and the ratchet member 154. Thus, a shift will not occur in this situation.

The terms of degree such as “substantially,” “about” and “approximately” as used herein mean a reasonable amount 40 of deviation of the modified term such that the end result is not significantly changed. These terms should be construed as including a deviation of at least ±5% of the modified term if this deviation would not negate the meaning of the word it modifies.

While only selected embodiments have been chosen to illustrate the present invention, it will be apparent to those skilled in the art from this disclosure that various changes and modifications can be made herein without departing 45 from the scope of the invention as defined in the appended claims. Furthermore, the foregoing description of the embodiments according to the present invention are provided for illustration only, and not for the purpose of limiting the invention as defined by the appended claims and their equivalents.

What is claimed is:

1. A bicycle shift operating device, comprising:
 - a first operating member;
 - a cable winding mechanism mounted on a first pivot axle 50 and configured to be selectively retained in one of a plurality of shift positions;

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an operating link having a first portion movably coupled to said first operating member and a second portion mounted on an auxiliary pivot axle, said auxiliary pivot axle being spaced from said first pivot axle and arranged to be parallel to said first pivot axle; and

a winding pawl coupled to said operating link and operatively engaged with said cable winding mechanism such that movement of said operating link about said auxiliary pivot axle in a first rotational direction rotates said cable winding mechanism about said first pivot axle between said shift positions in response to movement of said first operating member.

2. The bicycle shift operating device according to claim 1, wherein

said cable winding mechanism includes a cable winding member and a ratchet member mounted on said first pivot axle, and

a locking member is operatively engaged with said ratchet member to hold said cable winding member in one of said plurality of shift positions.

3. The bicycle shift operating device according to claim 2, wherein

said winding pawl is arranged to rotate said ratchet member upon movement of said first operating member from a normal rest position to a shift position.

4. The bicycle shift operating device according to claim 2, further comprising

a second operating member arranged to disengage said locking member from said ratchet member.

5. The bicycle shift operating device according to claim 2, wherein

said first operating member is pivotally coupled to a first end of a follower link that has a second end mounted on a second fixed pivot axle such that said first operating member, said operating link and said follower link form three links of a four bar linkage that controls movement of said first operating member.

6. The bicycle shift operating device according to claim 2, said cable winding member and said ratchet member are separate individual elements that are fixed to rotate together on said first pivot axle.

7. The bicycle shift operating device according to claim 2, wherein

said cable winding mechanism further includes a biasing member arranged to apply an urging force on said cable winding member and said ratchet member to rotate about said first pivot axle.

8. The bicycle shift operating device according to claim 1, wherein

said bicycle shift operating device is coupled to a handle bar mounting bracket that includes a brake lever.

9. The bicycle shift operating device according to claim 1, further comprising

a return biasing member arranged to operatively apply an urging force on said first operating member to urge said first operating member from a shift position to a normal rest position.

10. The bicycle shift operating device according to claim 1, wherein

said cable winding mechanism has more than three of said shift positions.